

WPI

Optimizing Partial Emission Pump Performance

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Our Team



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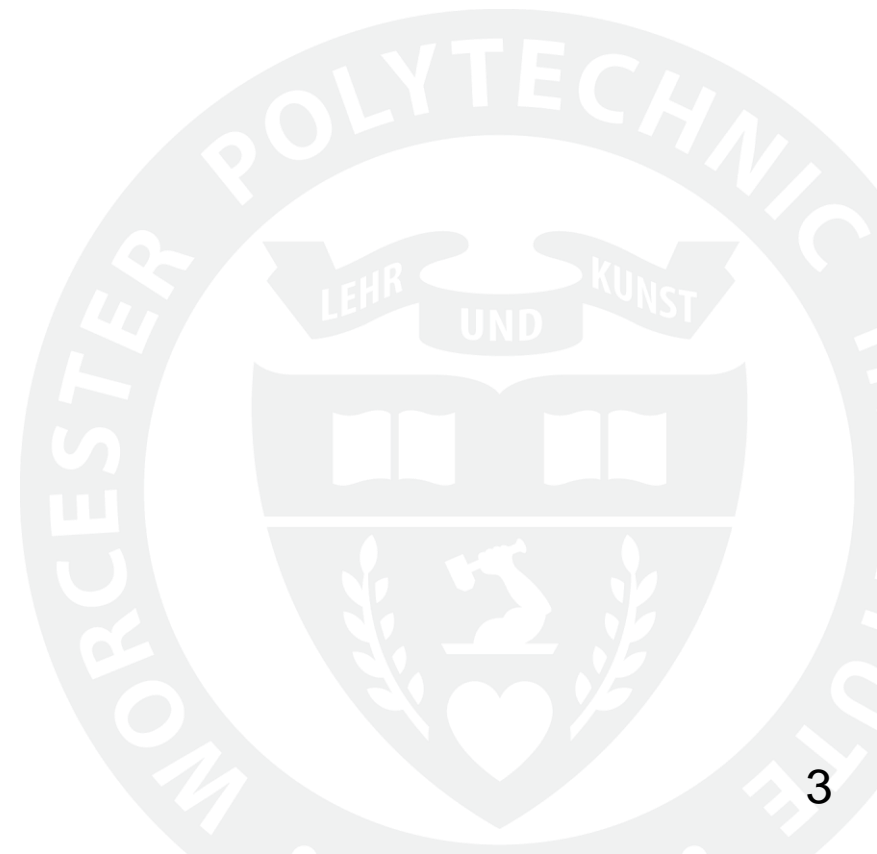
Shivaani Gopal
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Engineering



Keith Mesecher
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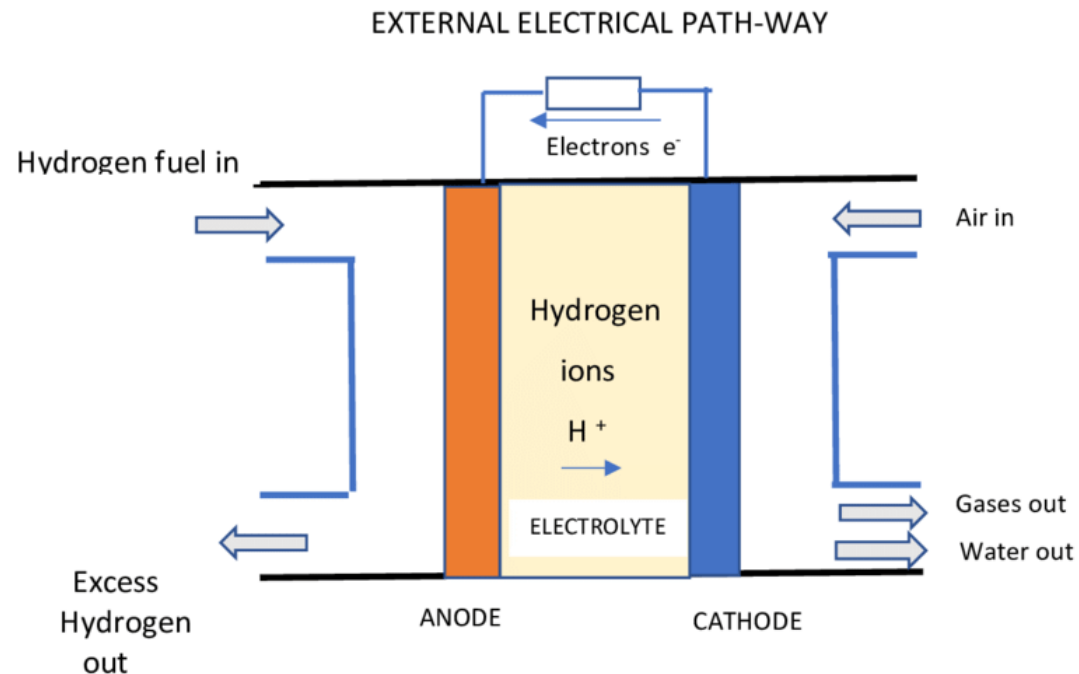
Introduction

What problem is our project addressing?



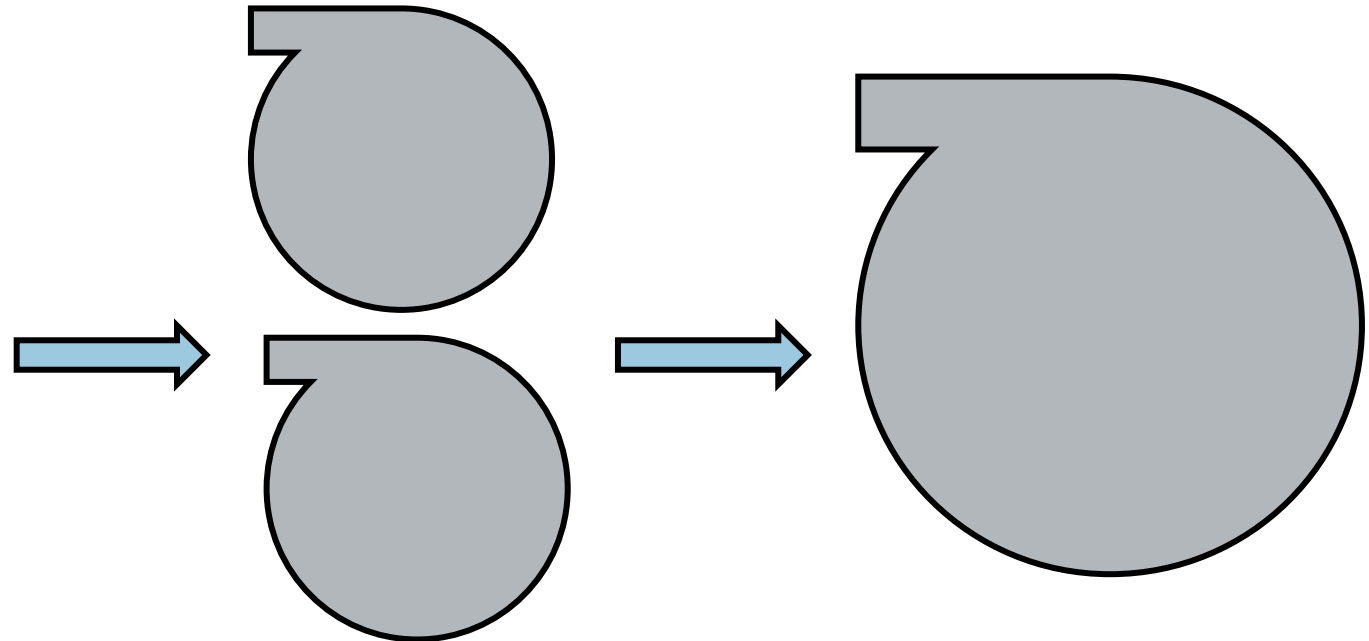
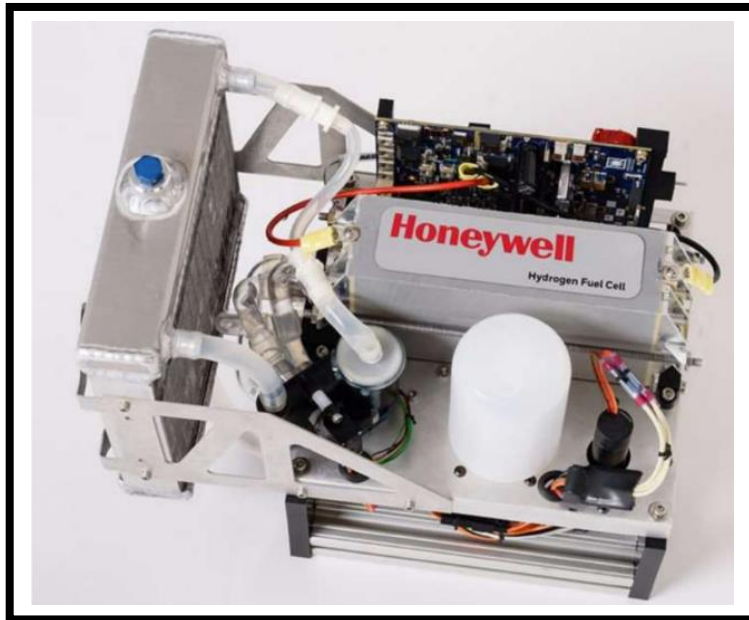
What is Hydrogen Fuel Cell Technology?

- Alternative cleaner energy source
- Electricity is generated through an electrochemical reaction



Project Problem

- Double the 600 watt hydrogen fuel cell stack
- Optimize a large pump to account for the total 1200 watt stack

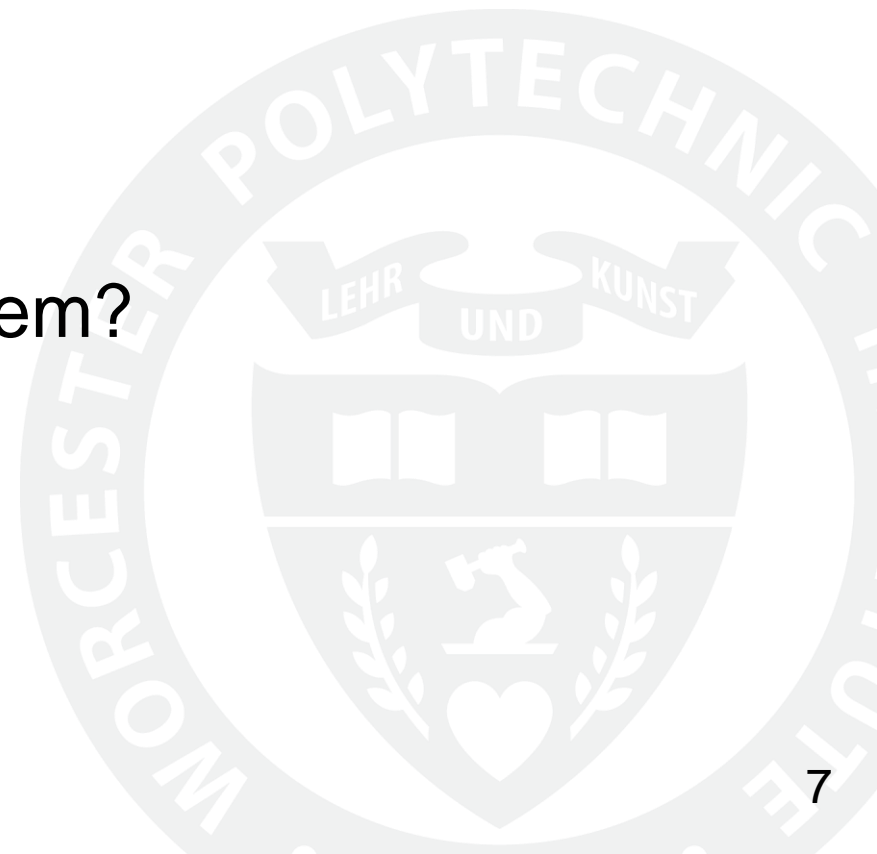


Problem Definition

Optimize PEM pump performance for scaling hydrogen fuel cell systems to power larger UAVs efficiently.

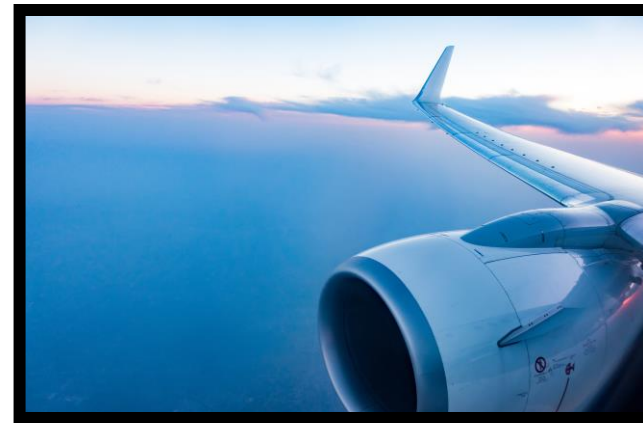
Background

What are PEM pumps? How do we analyze them?



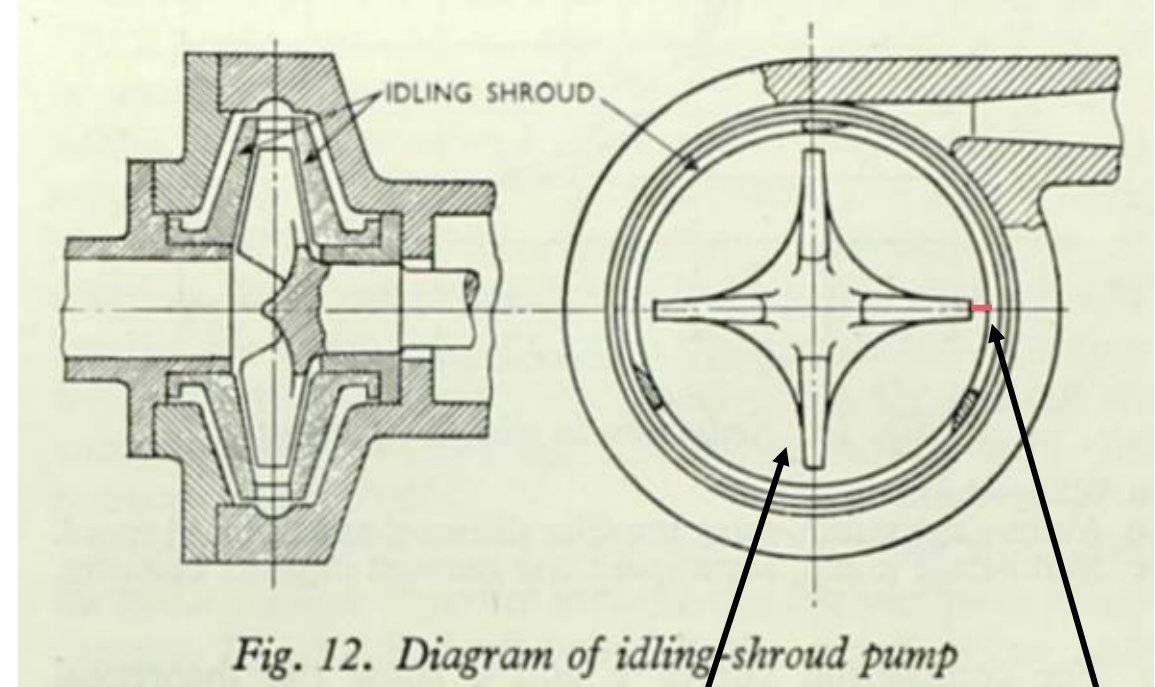
What are Pumps?

- Mechanical systems:
 - Uses energy to move fluid → affects several output parameters
- How they work:
 - Uses a mechanism (i.e. impeller) to drive fluid from inlet to outlet
- Used in several applications:



PEM Pump Research

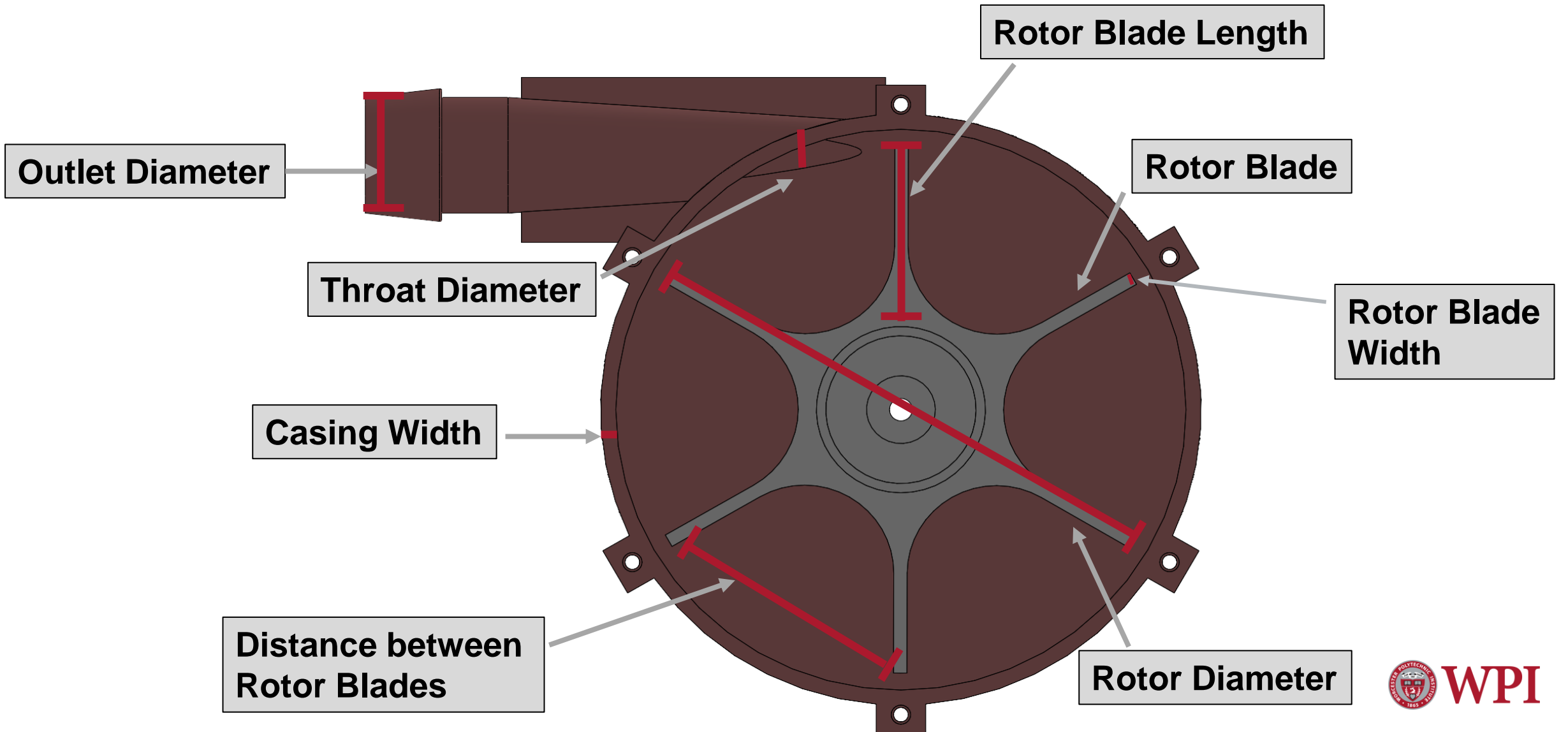
- Centrifugal Pumps → PEM Pumps
- Produce low flow and high head
- Development is credited to U.M. Barske
- Limited research on PEM Pumps



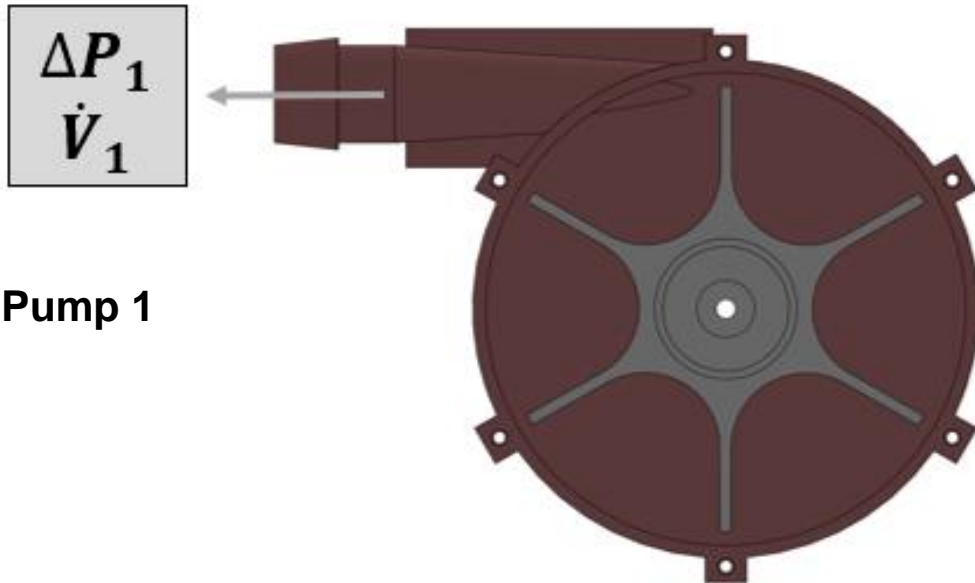
Straight Impeller
Blade

Vane Size
Gap

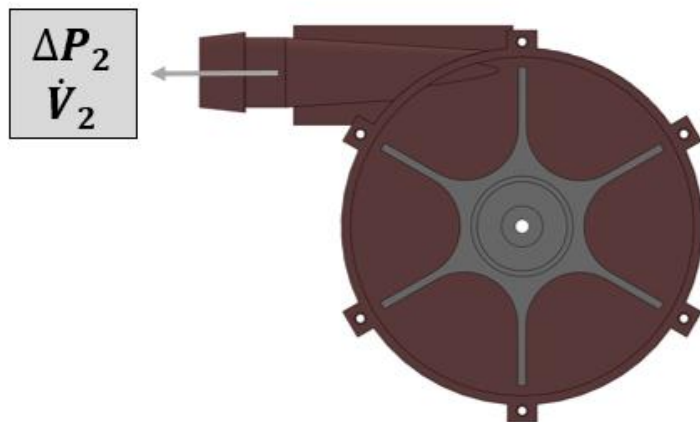
PEM Diagram



Nondimensional Variables

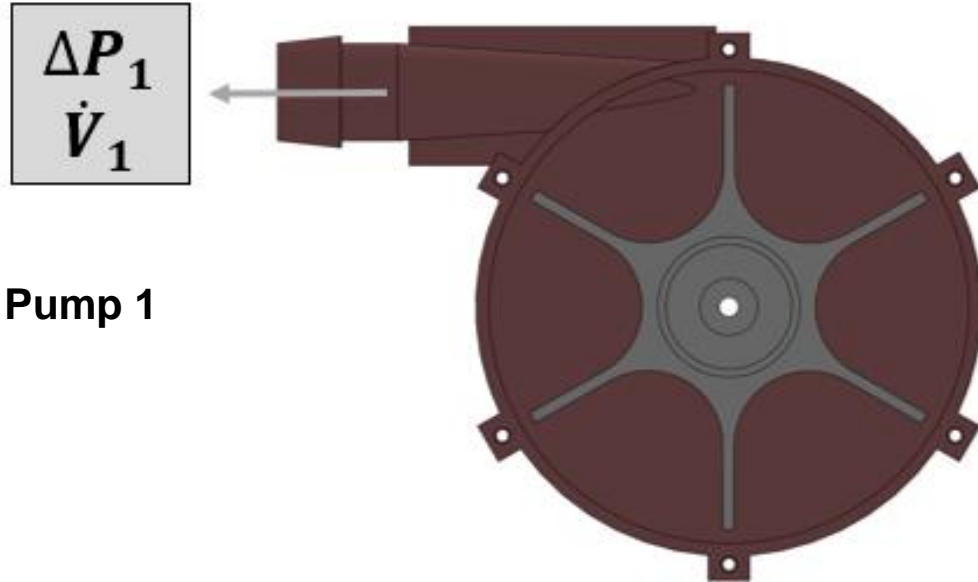


Pump 1

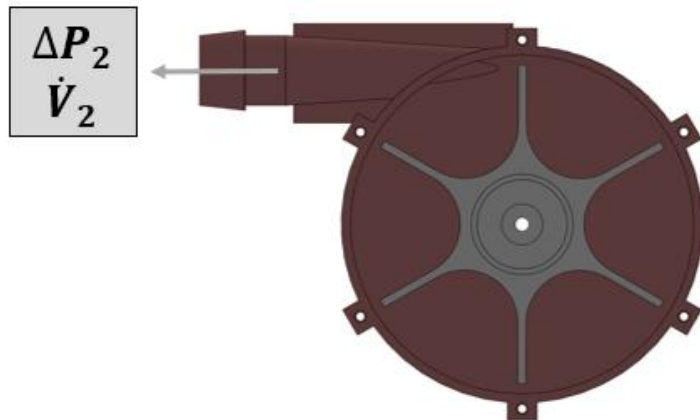


Pump 2

Nondimensional Variables



Pump 1

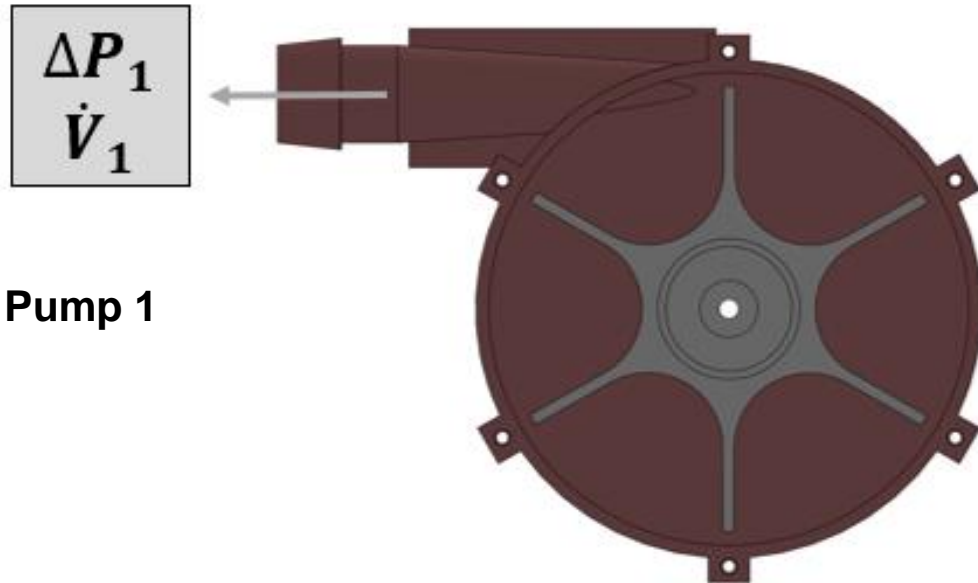


Pump 2

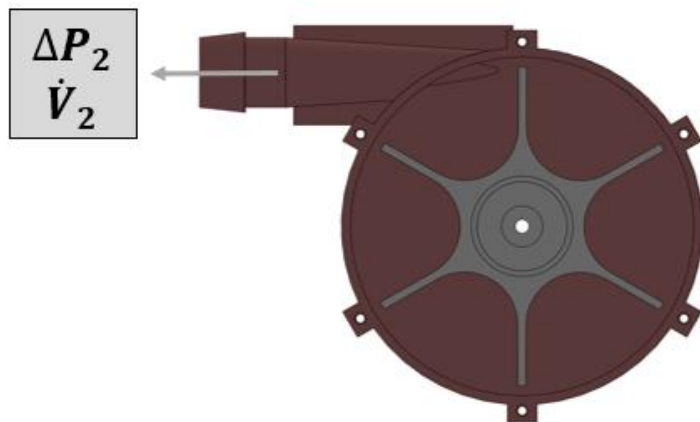
Dimensional Parameters

$$\omega_{rotor} \quad D_{rotor}$$
$$D_{throat} \quad \rho_{fluid}$$

Nondimensional Variables



Pump 1



Pump 2

Dimensional Parameters

$$\omega_{rotor} \quad D_{rotor}$$
$$D_{throat} \quad \rho_{fluid}$$



Nondimensional Parameters

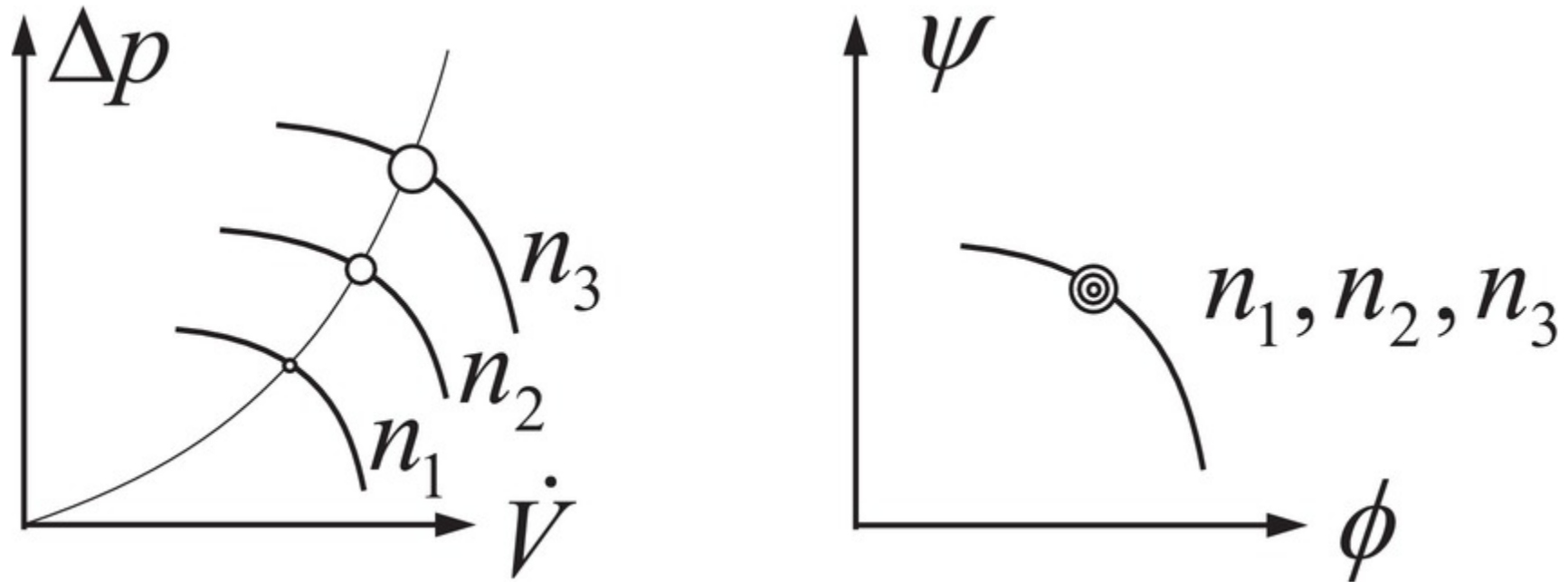
$$\phi, \varphi$$

OR

$$n_s, d_s$$

Nondimensional Variables

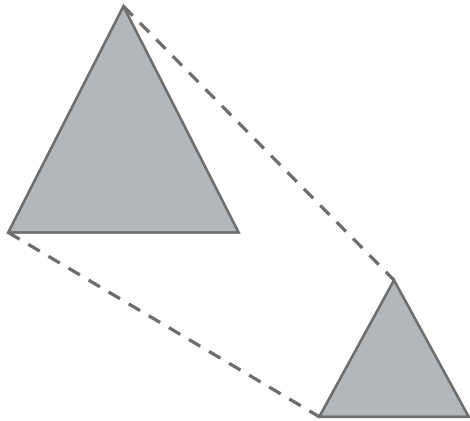
- Dimensional Curves collapse onto one dimensionless curve



Similarity Requirements

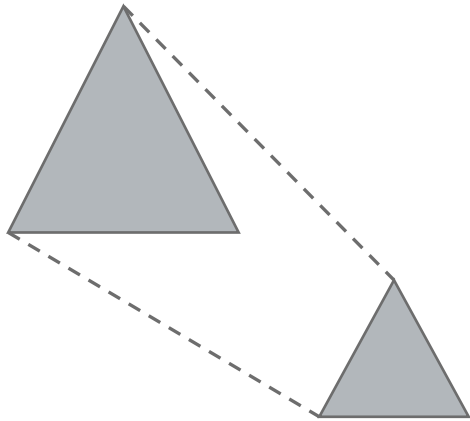
Similarity Requirements

Geometric Similarity

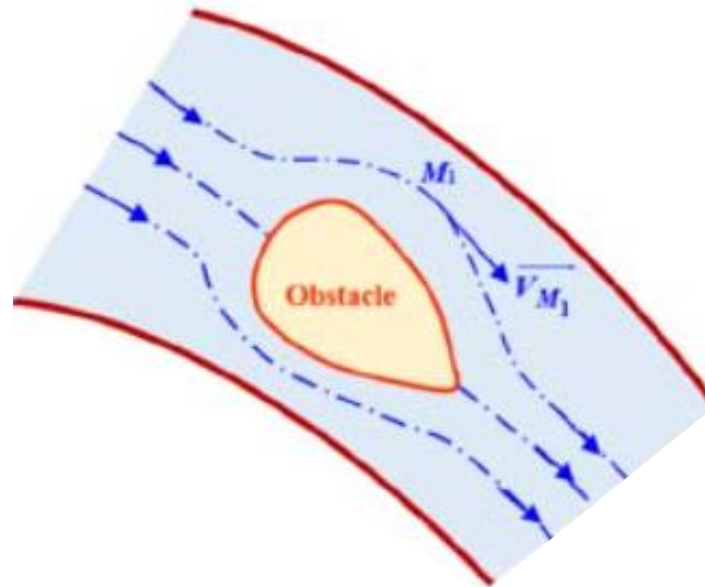


Similarity Requirements

Geometric Similarity

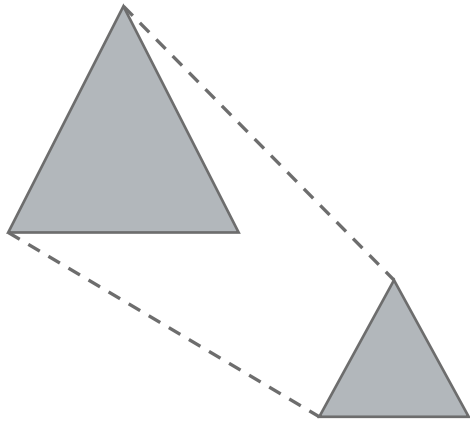


Fluid Dynamic Similarity

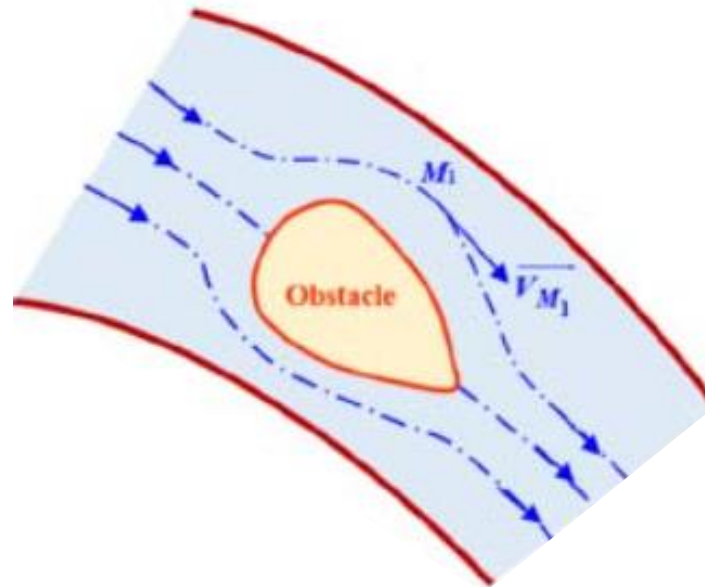


Similarity Requirements

Geometric Similarity



Fluid Dynamic Similarity



Thermodynamic Similarity

$$PV = nRT$$

Specific Speed and Specific Diameter

Specific Speed and Specific Diameter

Specific Speed

$$n_s = \frac{\omega \sqrt{\dot{V}}}{(Hg)^{3/4}}$$

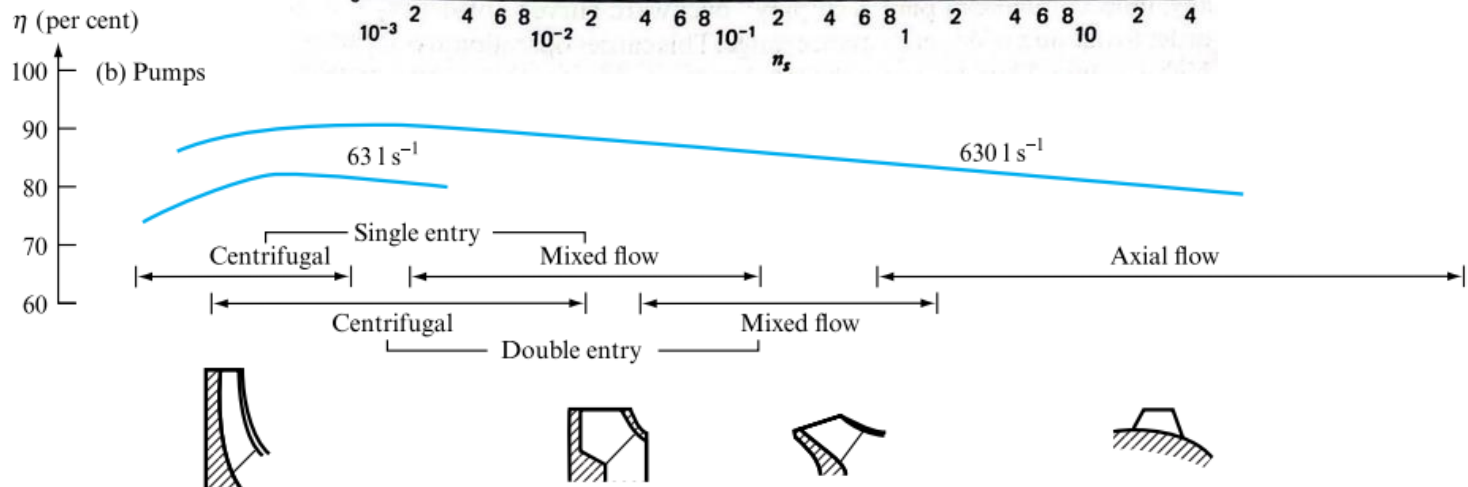
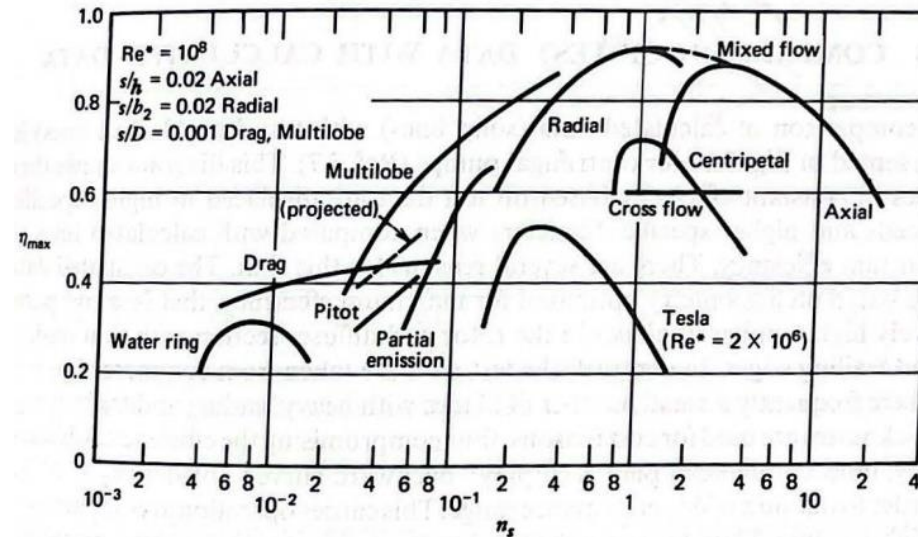
The **rotational speed** at which a geometrically similar pump would operate if it were delivering **one unit of flow rate** and developing **one unit of head**.

Specific Speed and Specific Diameter

Specific Speed

$$n_s = \frac{\omega \sqrt{\dot{V}}}{(Hg)^{3/4}}$$

The **rotational speed** at which a geometrically similar pump would operate if it were delivering **one unit of flow rate** and developing **one unit of head**.



Specific Speed and Specific Diameter

The **rotor diameter** of a geometrically similar pump that would deliver **one unit of flow rate** while developing **one unit of head**.

Specific Diameter

$$d_s = \frac{D_{rotor}(Hg)^{1/4}}{\sqrt{\dot{V}}}$$

Specific Speed and Specific Diameter

Specific Speed

$$n_s = \frac{\omega \sqrt{\dot{V}}}{(Hg)^{3/4}}$$

Specific Diameter

$$d_s = \frac{D_{rotor} (Hg)^{1/4}}{\sqrt{\dot{V}}}$$

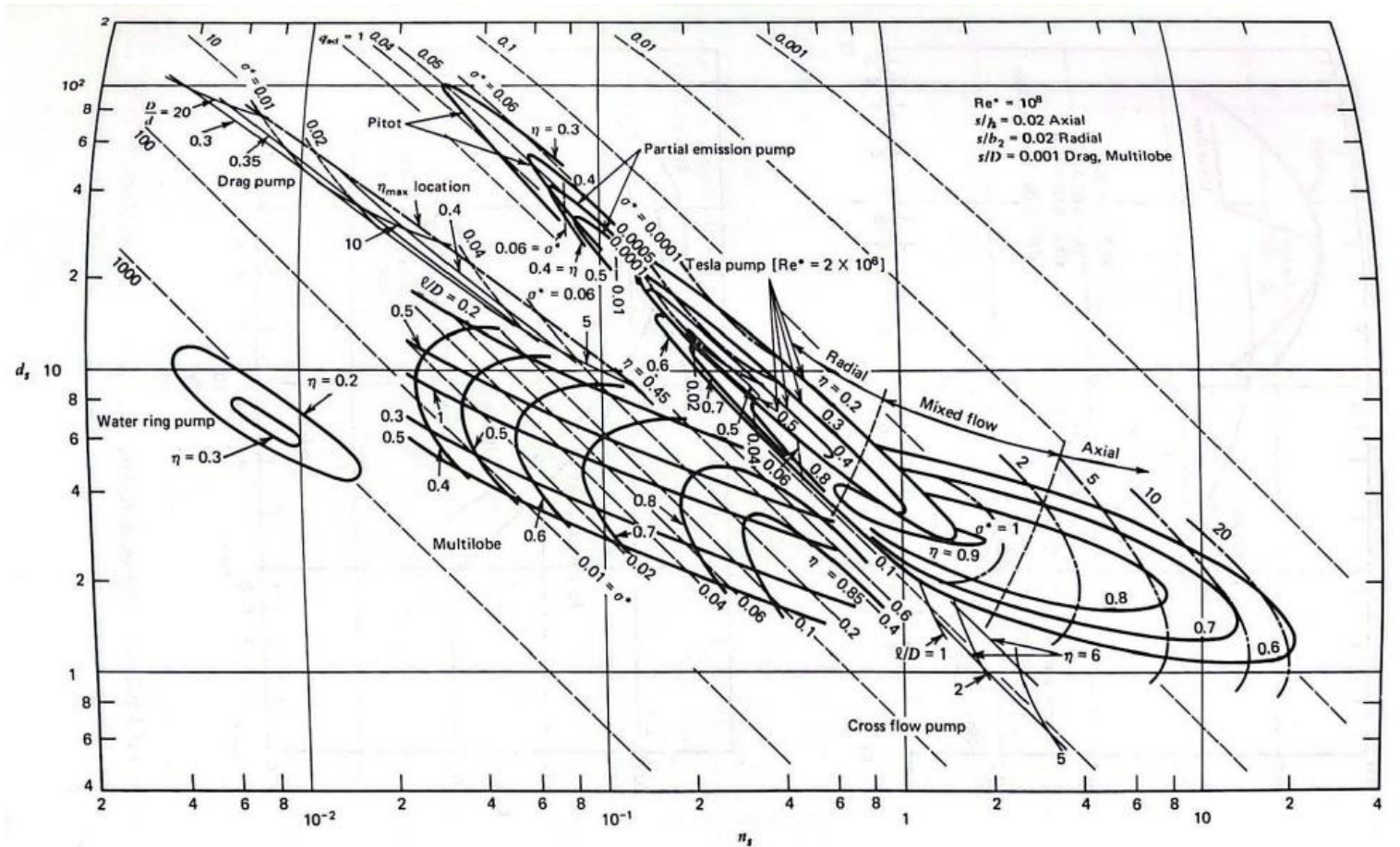


FIGURE 3.8 $n_s d_s$ diagram for single stage pumps.

Specific Speed and Specific Diameter

Specific Speed

$$n_s = \frac{\omega \sqrt{\dot{V}}}{(Hg)^{3/4}}$$

Specific Diameter

$$d_s = \frac{D_{rotor} (Hg)^{1/4}}{\sqrt{\dot{V}}}$$

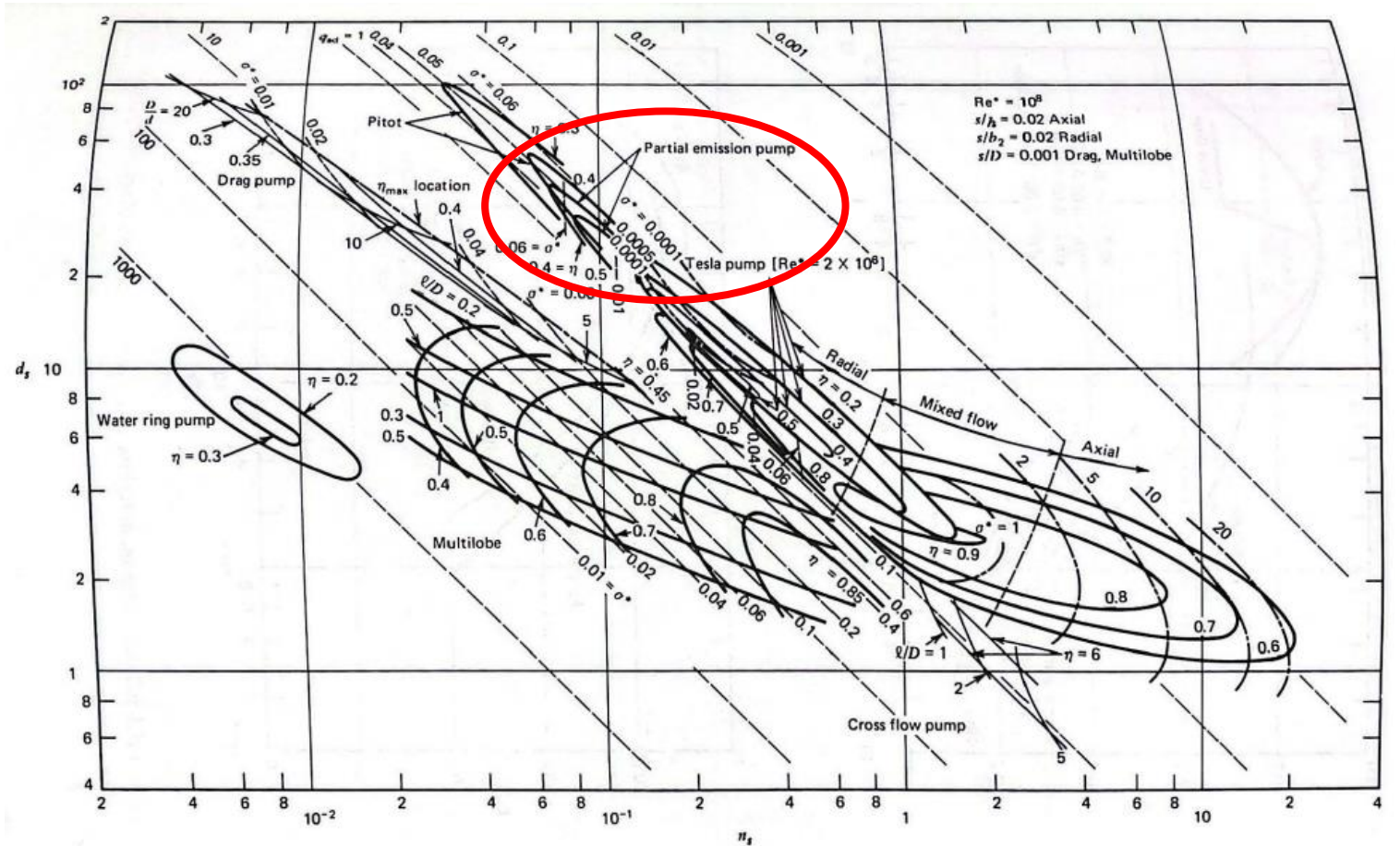


FIGURE 3.8 $n_s d_s$ diagram for single stage pumps.

Pressure and Flow Coefficients

Pressure and Flow Coefficients

$$\textit{Pressure Coefficient} = \psi = \frac{\Delta p / \rho}{U^2}$$

The **ratio** of the **pump pressure rise** to the **dynamic pressure of the fluid flow**.

$$U_2 = \pi \cdot n \cdot D_{\text{rotor}}$$

Pressure and Flow Coefficients

$$\text{Pressure Coefficient} = \psi = \frac{\Delta p / \rho}{U^2}$$

The **ratio** of the **pump pressure rise** to the **dynamic pressure of the fluid flow**.

$$\text{Flow Coefficient} = \phi = \frac{v_{out}}{U}$$

The **ratio** of the **throughput component of the velocity** to the **blade tip speed**.

$$U_2 = \pi \cdot n \cdot D_{rotor} \quad v_{out} = \frac{\dot{V}}{\frac{1}{4} \pi \cdot D_{diffuser}^2}$$

Pressure and Flow Coefficients

$$\text{Pressure Coefficient} = \psi = \frac{\Delta p / \rho}{U^2}$$

$$\text{Flow Coefficient} = \phi = \frac{v_{out}}{U}$$

$$U_2 = \pi \cdot n \cdot D_{rotor} \quad v_{out} = \frac{\dot{V}}{\frac{1}{4} \pi \cdot D_{diffuser}^2}$$

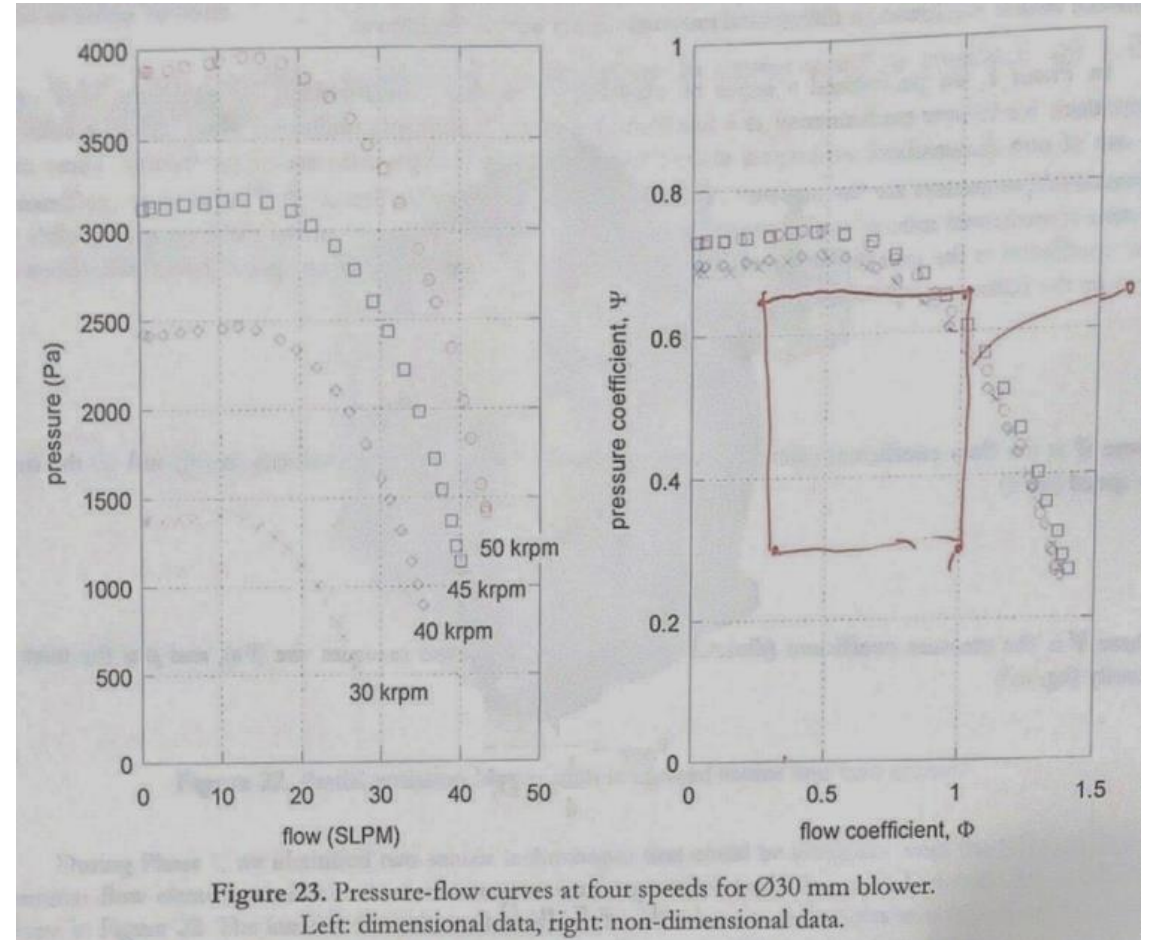


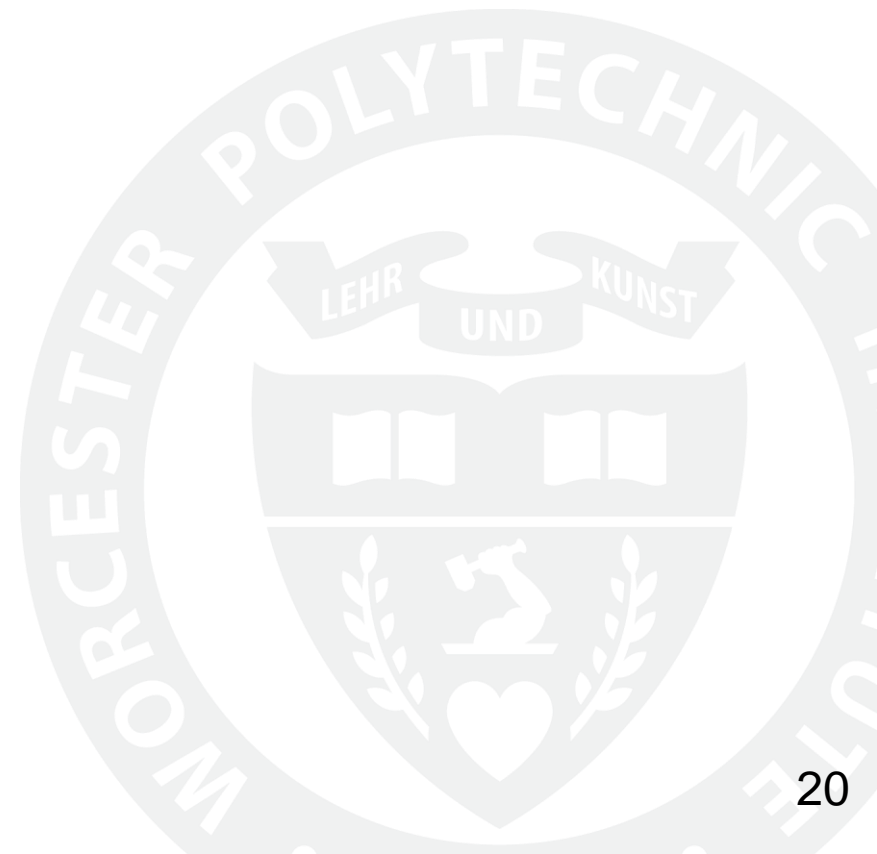
Figure 23. Pressure-flow curves at four speeds for Ø30 mm blower.
Left: dimensional data, right: non-dimensional data.

Project Objective

Develop an **empirical model** that accurately **captures the performance characteristics of PEM pumps** under diverse operating conditions.

Methodology

How did we reach this objective?



Methodology Overview

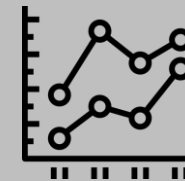
Build geometrically similar pumps



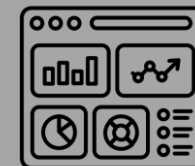
Build experimental test setup



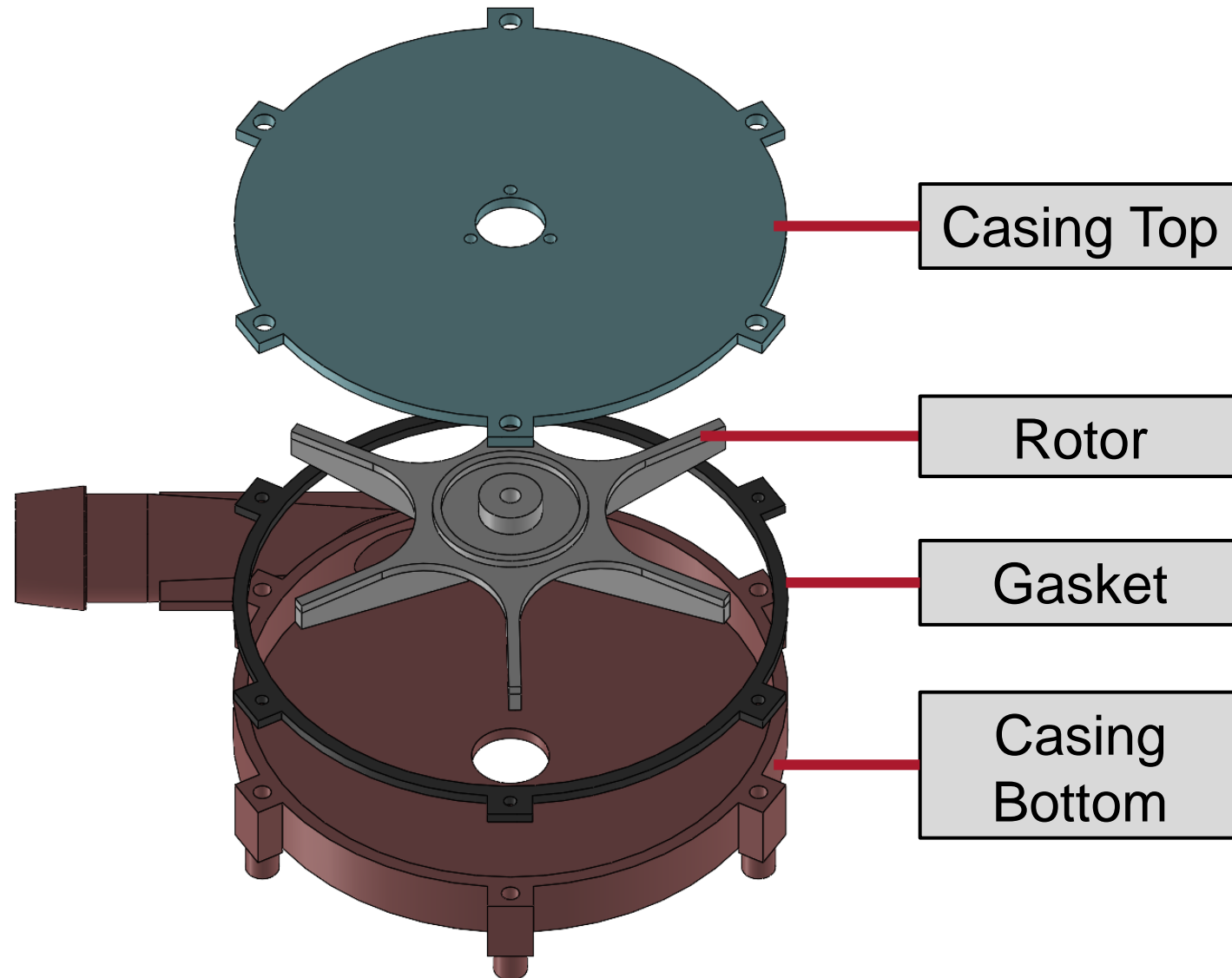
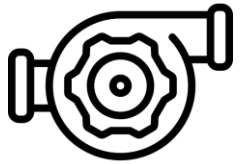
**Collect experimental data to formulate
nondimensional variables**



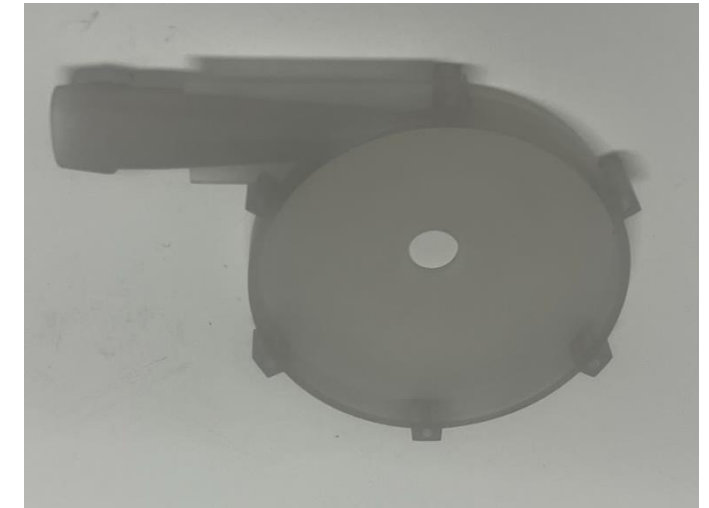
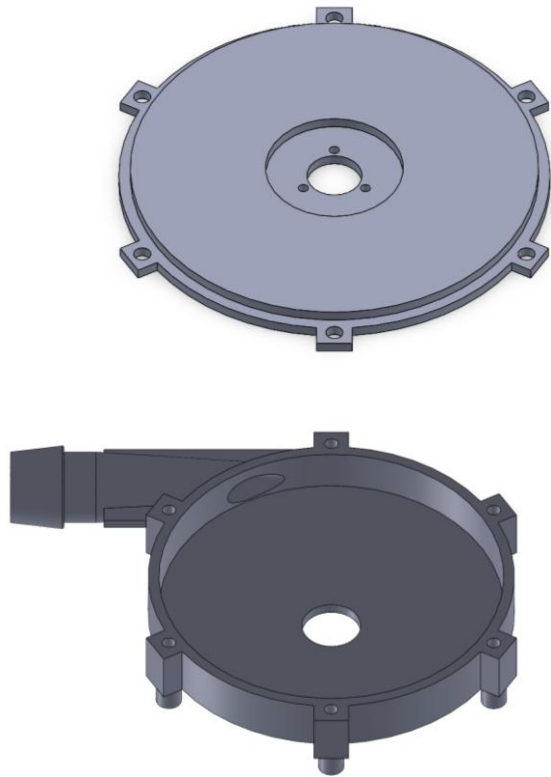
**Create empirical model with
experimental data**



Building the Pumps



Casing Design and Manufacturing

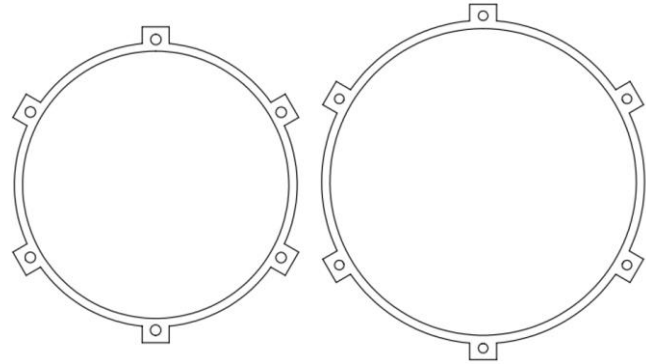
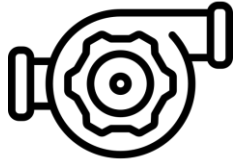


Models for each rotor size were created in CAD software

Prototypes were 3D printed in PLA

Final casings were 3D printed in resin

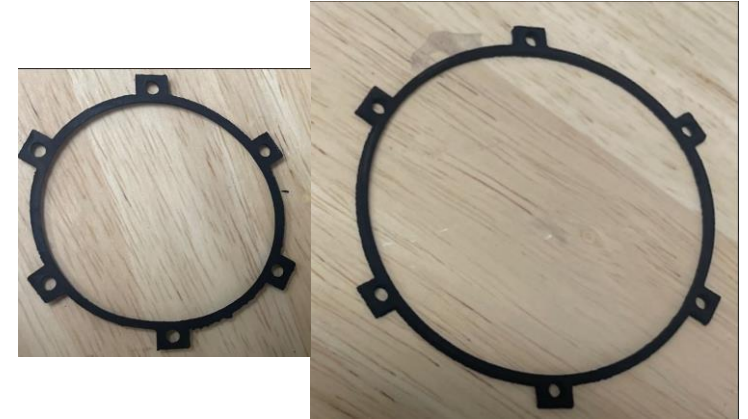
Gasket Design and Manufacturing



Drawings were created in CAD software

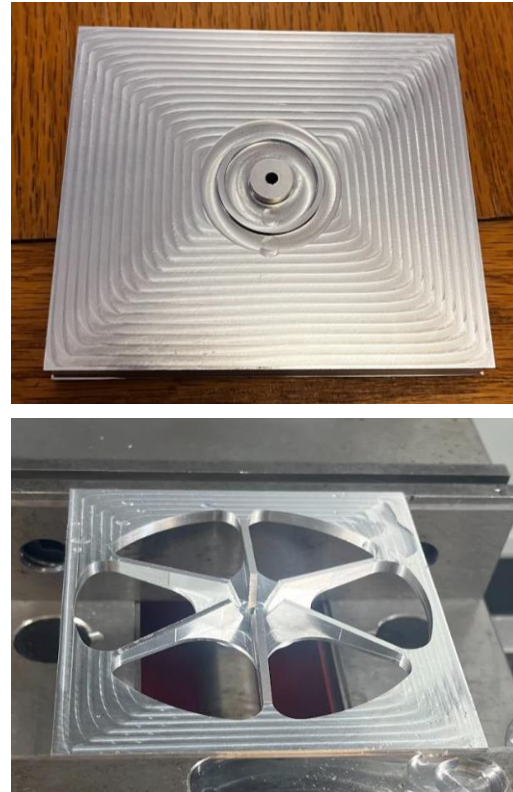
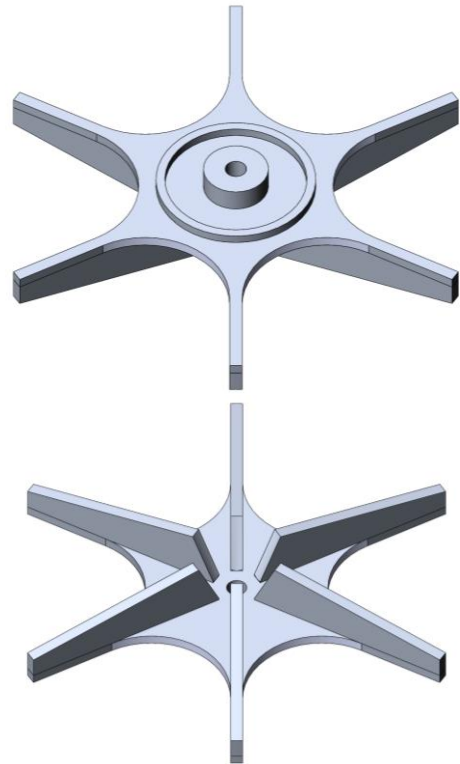
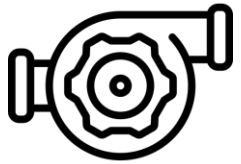


Drawings were uploaded to laser cutter and gaskets were cut out of rubber sheets



Finished gaskets for 40 and 60 mm diameter pumps

Rotor Design and Manufacturing (70 mm)

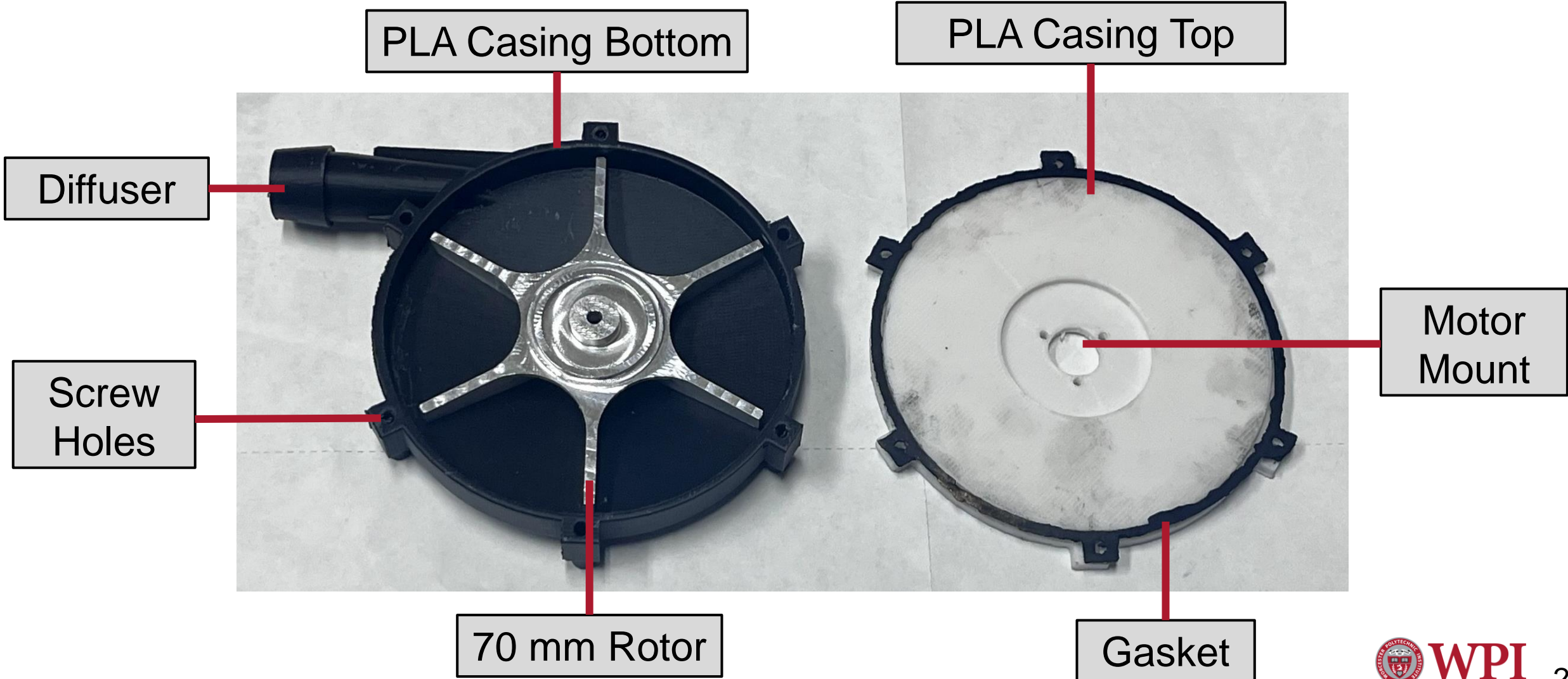
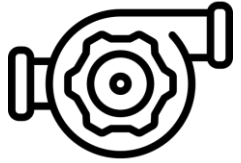


Blades extended to 70 mm & CAM developed for both sides

Both sides were CNC machined out of aluminum stock

Rotor was cut out of stock & sanded

Prototype Pump Assembly



PLA Casing Bottom

PLA Casing Top

Diffuser

Screw Holes

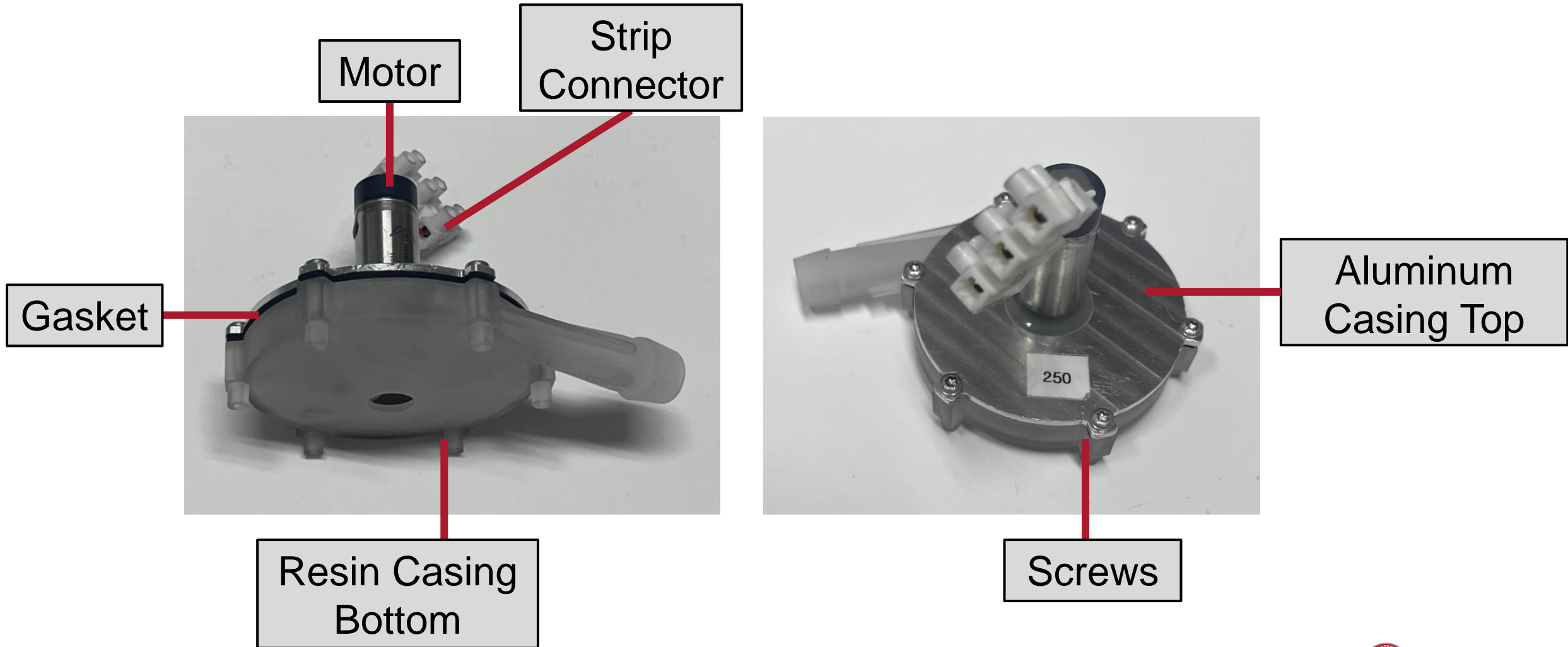
70 mm Rotor

Motor Mount

Gasket

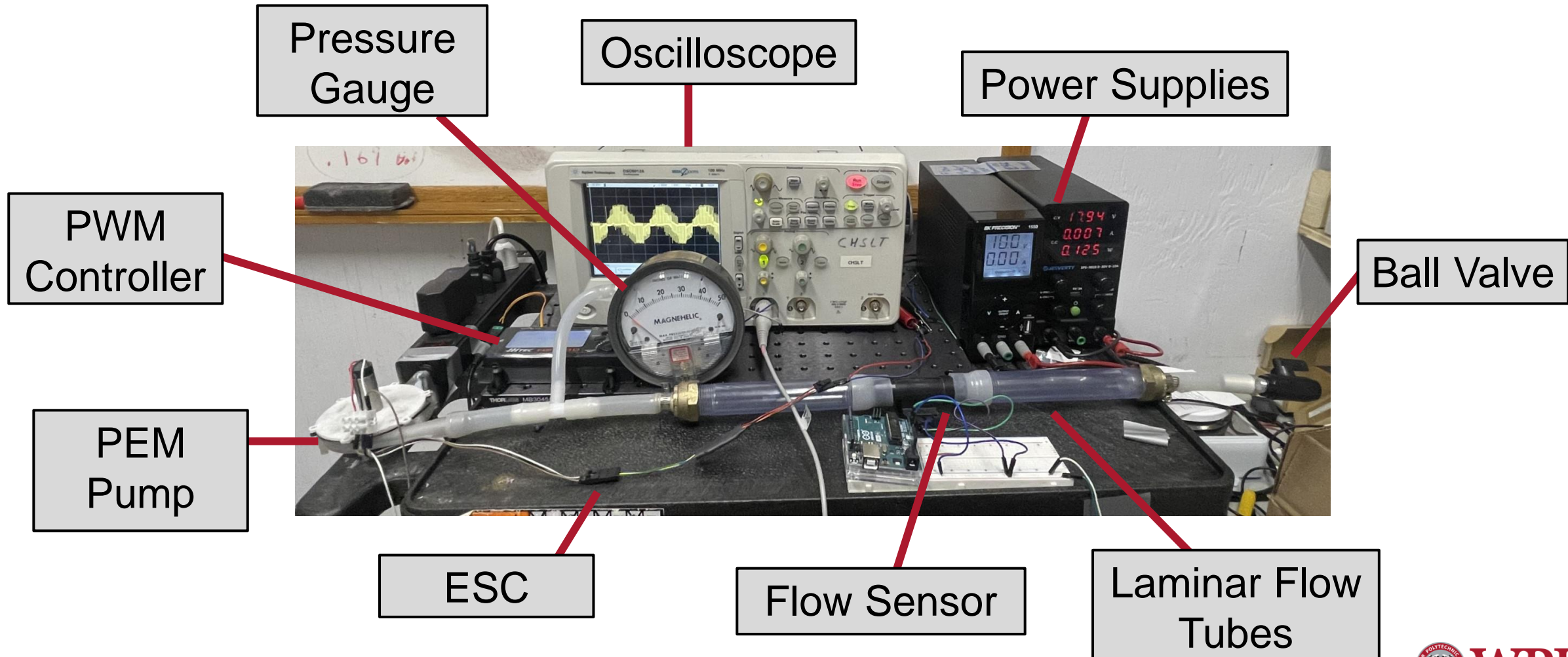


Final Pump Assembly with Motor





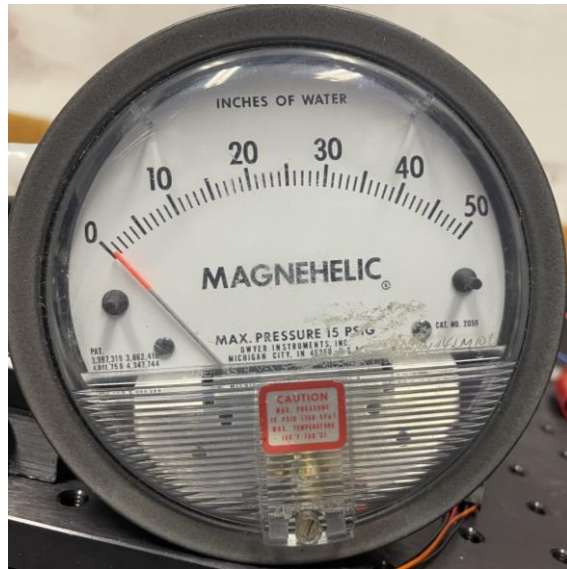
Experimental Setup



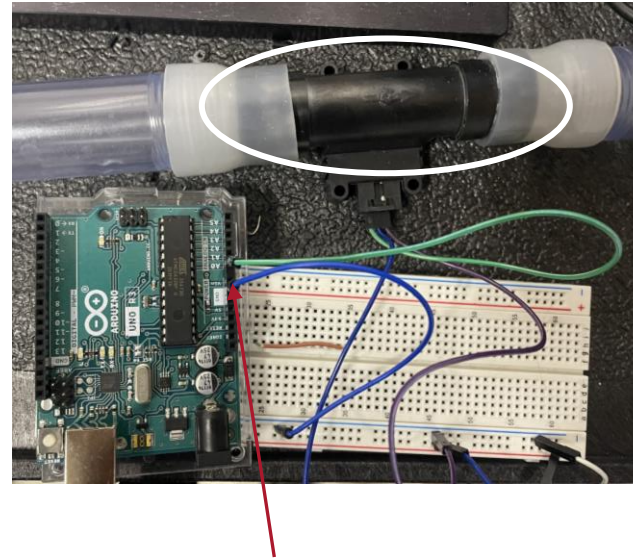


Measuring Parameters

Pressure

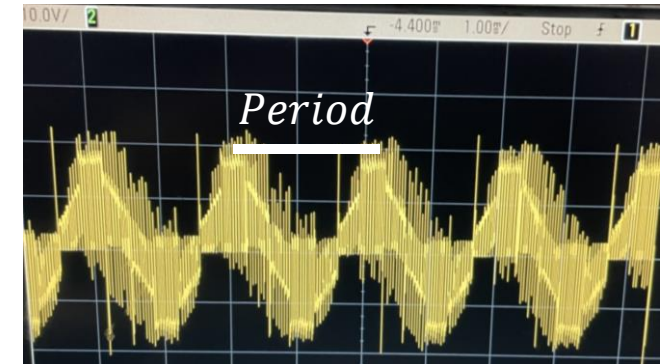


Flow



Flow = f(voltage)

Rotational Speed



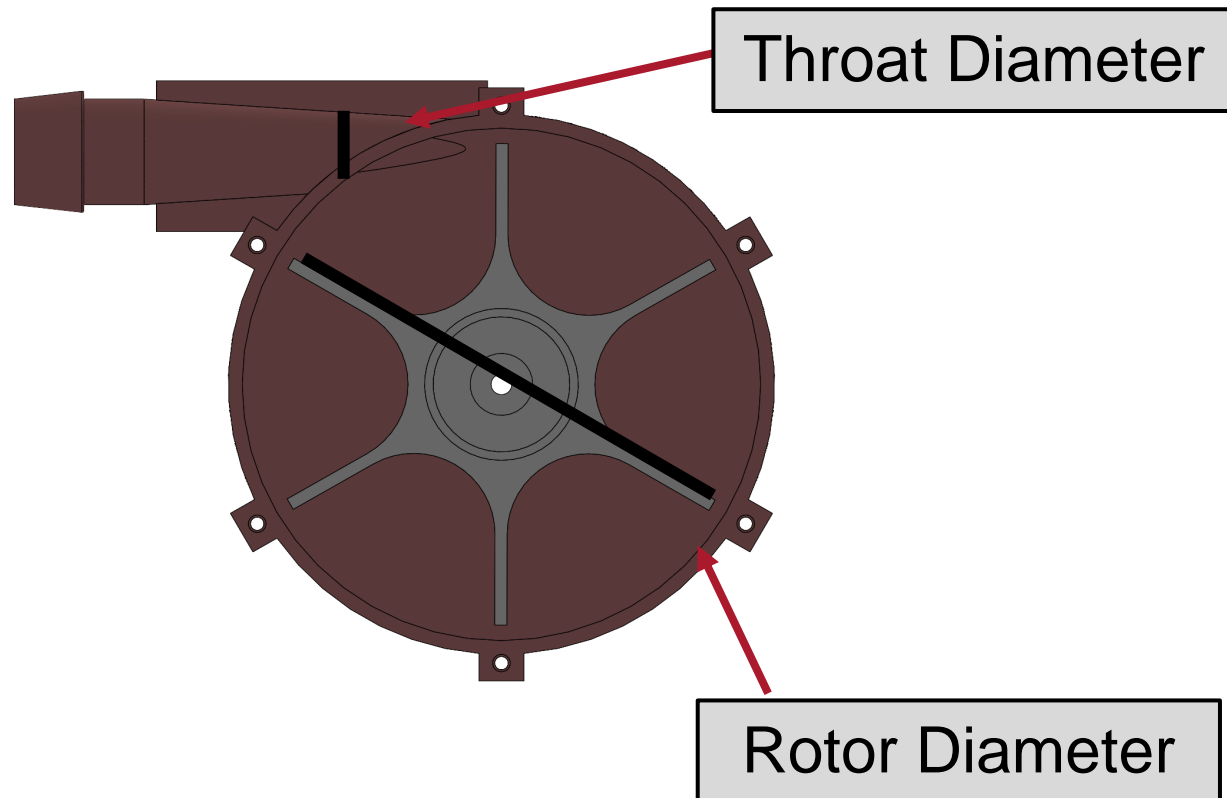
$$\text{Frequency } f_s = \frac{1}{\text{Period}}$$

$$\omega = \frac{120f_s}{\text{no. of poles}}$$



Measuring Rotor Parameters

Rotor and Throat Diameter



Power

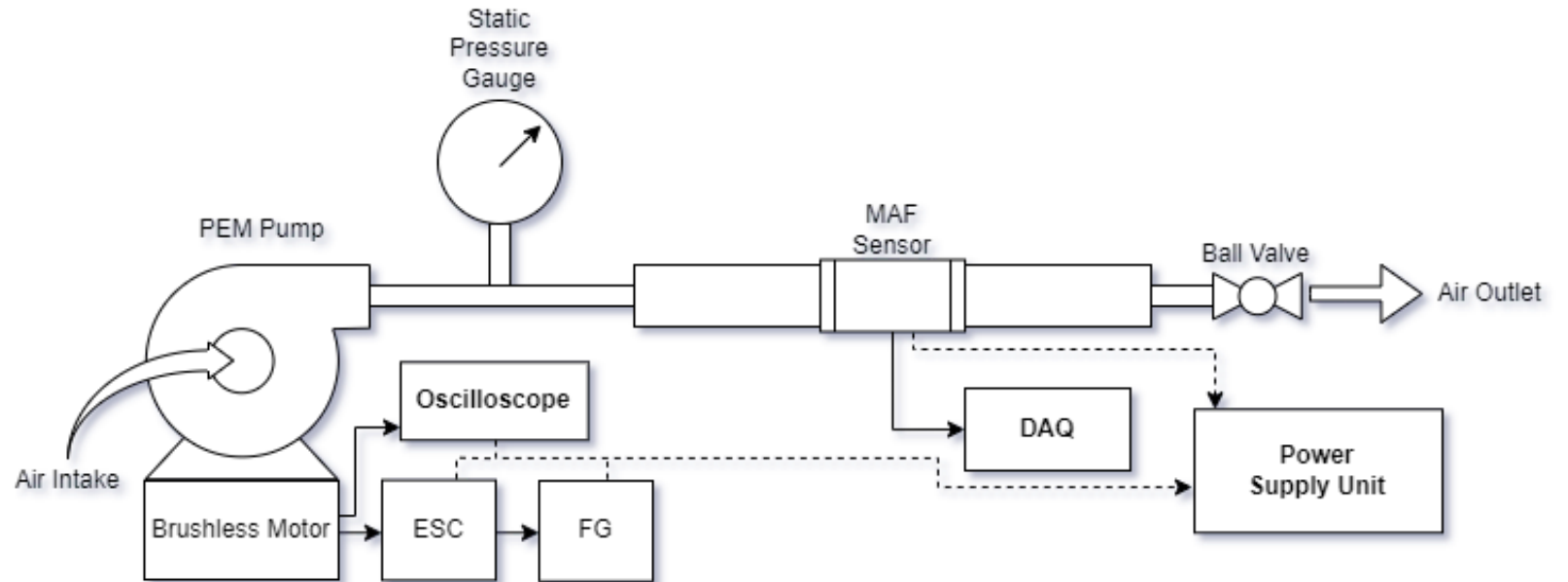




Experimental Setup

- Parameters Measured:

- Pressure
- Flow
- Rotor Speed
- Throat Diameter
- Rotor Diameter
- Power



Creating Non-Dimensional Values



Pressure Rise (Pa)

Rotor Speed (RPM)

Rotor Diameter (m)

Throat Diameter (m)

Flow (SLPM)



$$\psi = \frac{\Delta p / \rho}{U^2}$$

$$\phi = \frac{v_{out}}{U}$$



$$\psi = f(\phi)$$

$$\phi = f(\psi)$$



$$d_s = \frac{D_{rotor} (Hg)^{1/4}}{\sqrt{\dot{V}}}$$

$$n_s = \frac{\omega \sqrt{\dot{V}}}{(Hg)^{3/4}}$$



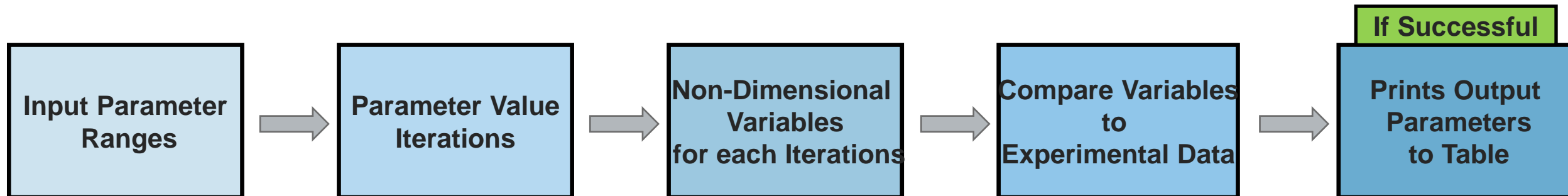
$$d_s = f(n_s)$$

$$n_s = f(d_s)$$

Model Creation

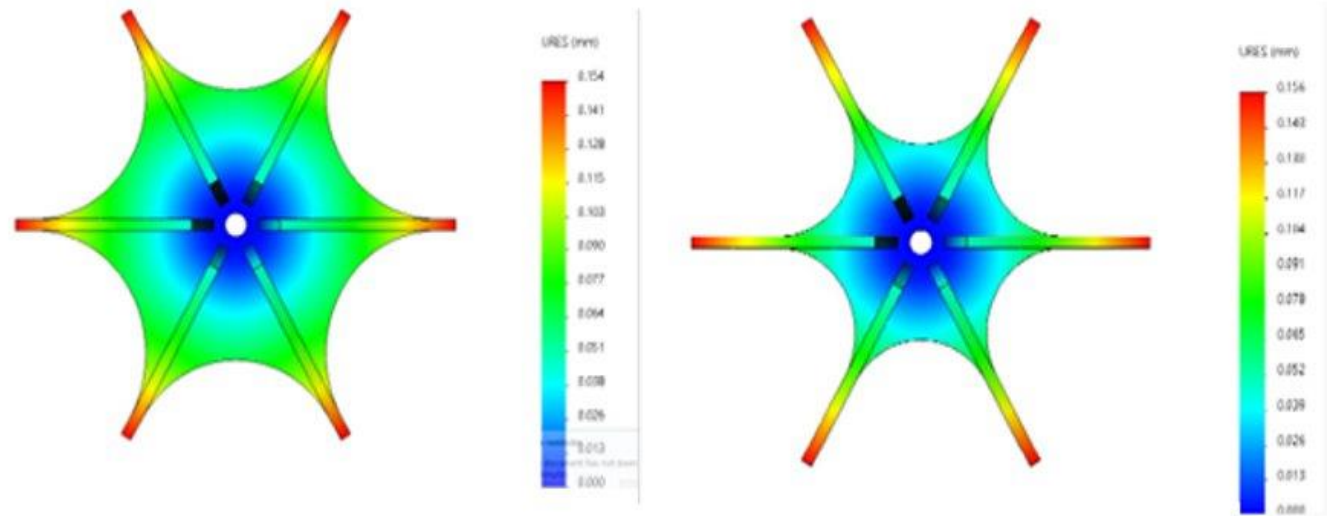


- An empirical model on MATLAB was developed to optimize the desired parameters of the pump
- A curve fit was created from experimental pressure and flow coefficients to develop equations for an output value from the input values



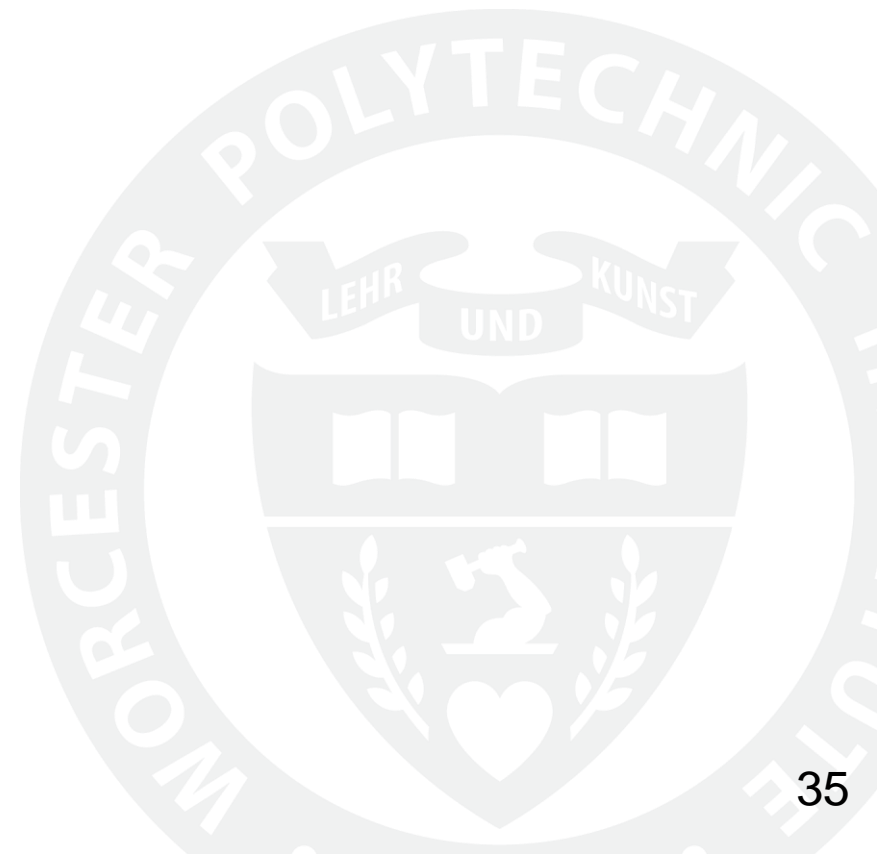
Rotor Deflection

- Blades of non-metal impellers experience a deflection effect from centrifugal forces
- Simulations were developed in SolidWorks and used to test different impeller parameters



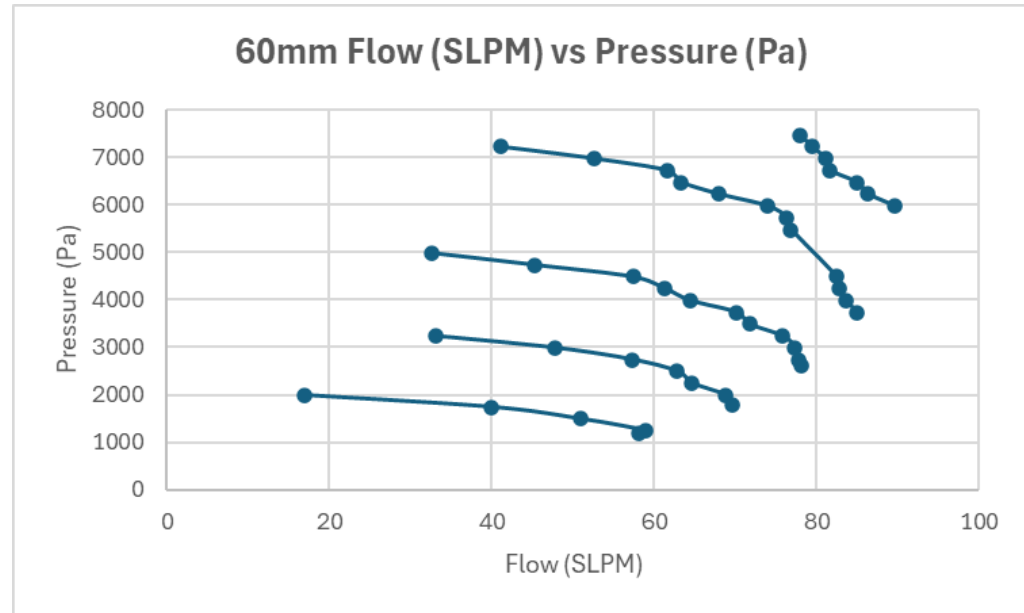
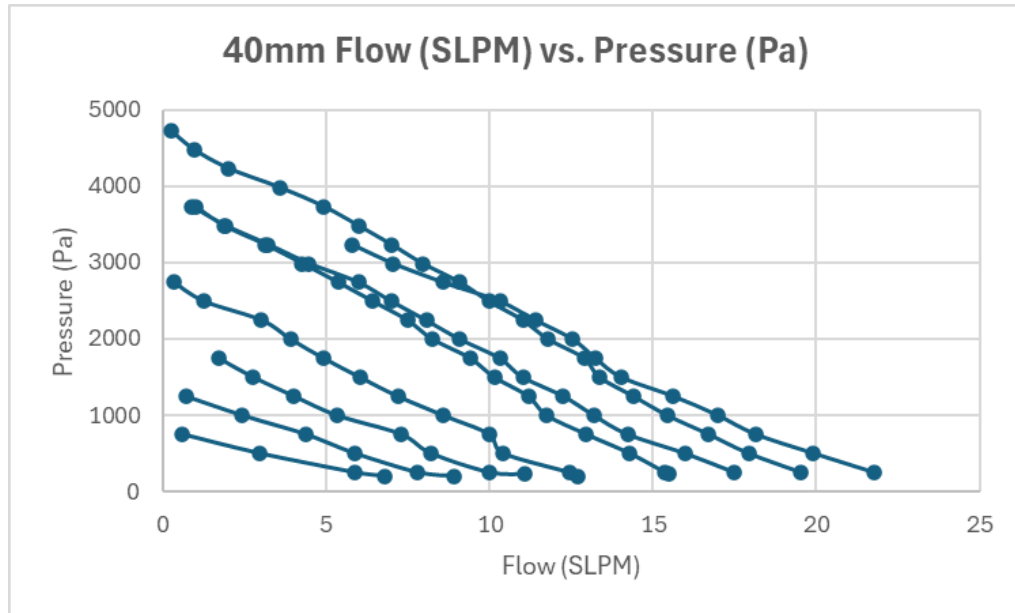
Results

What did we find?



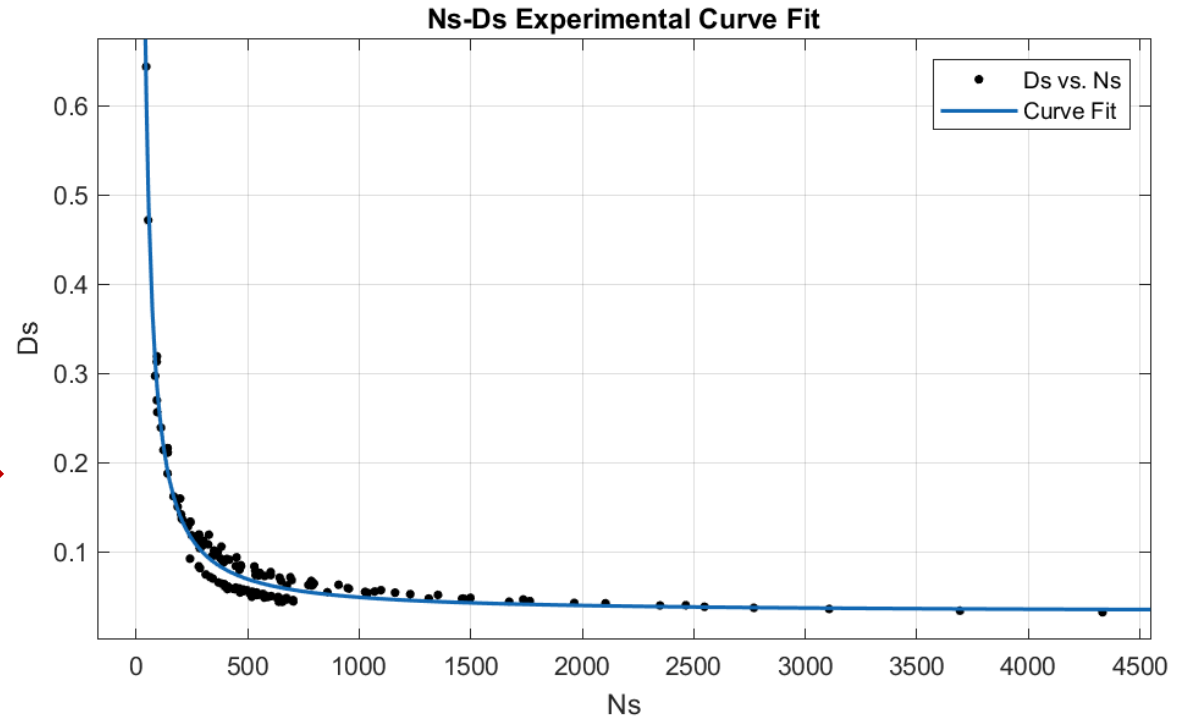
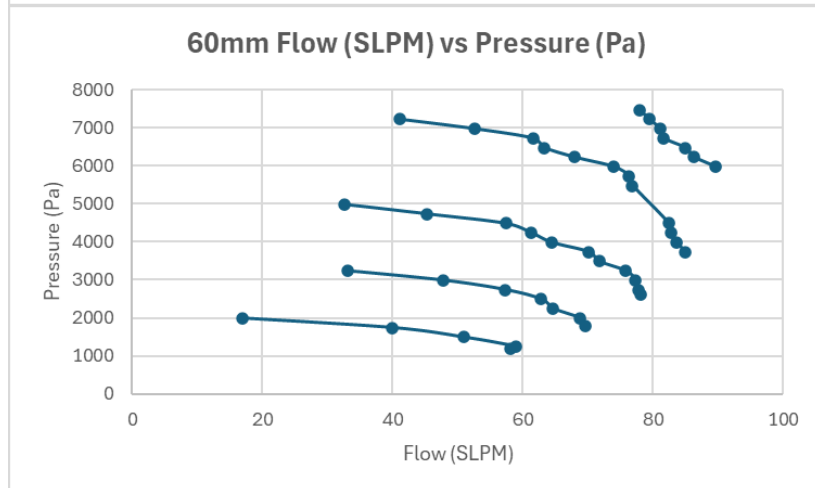
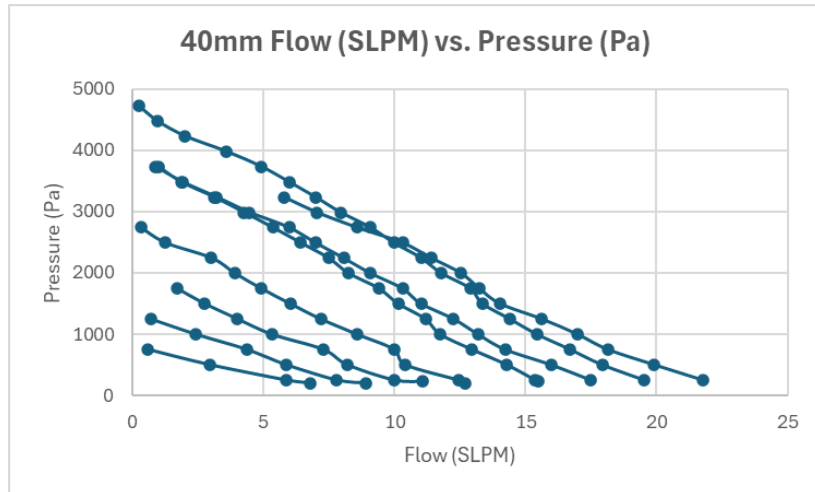
Experimental Setup Results

- Measured performances of 40 mm, 60mm PEM pumps



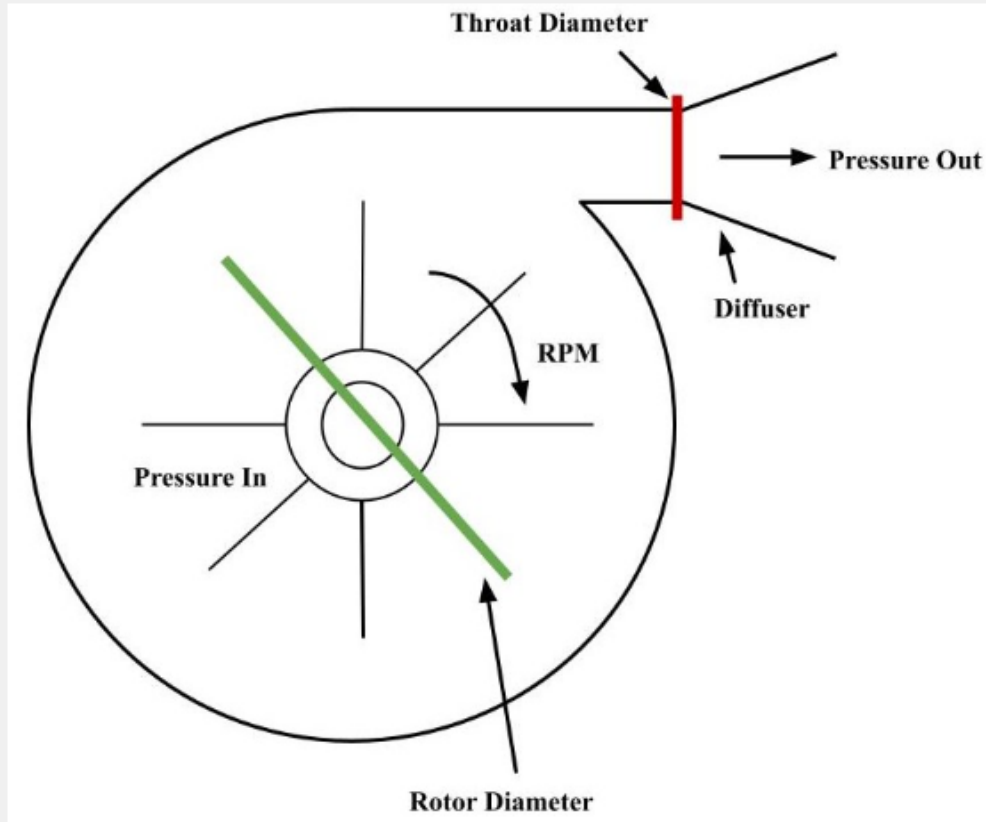
Experimental Setup Results

- Developed non-dimensional curve fits based off experimental results



$$D_s = 46.308N_s^{-1.1473} + 0.0324$$

Partial Emissions Blower Model



Mass Flow Rate Min Max Exact

Rotor Diameter Min Max Exact

Throat Diameter Min Max Exact

Rotational Speed Min Max Exact

Pressure Rise Min Max Exact

Fluid: Iterations:

Calculate

0 % Completion

Successes

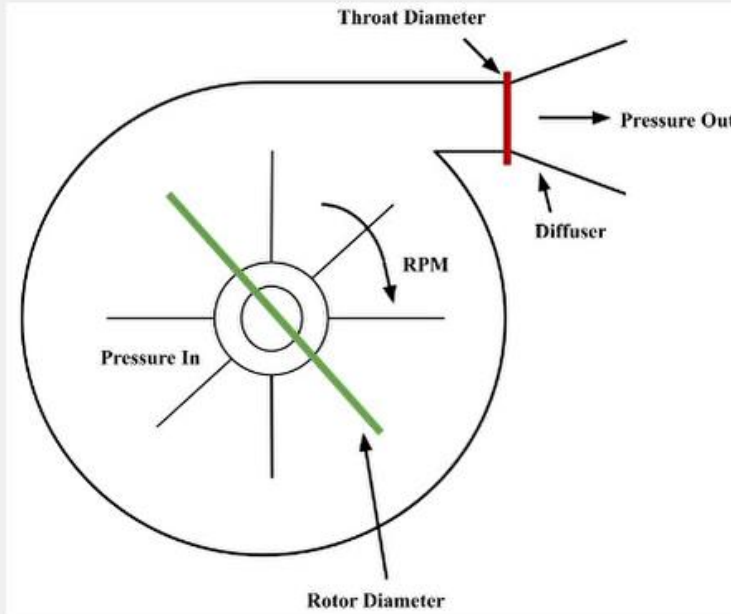
Elapsed Time (s)

Stop

Pressure Rise	RPM	Rotor Diameter	Throat Diameter	Mass Flow Rate	Fluid Density
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Model Features

Partial Emissions Blower Model



Mass Flow Rate Min Max Exact

Rotor Diameter Min Max Exact

Throat Diameter Min Max Exact

Rotational Speed Min Max Exact

Pressure Rise Min Max Exact

Fluid: Iterations:

Calculate

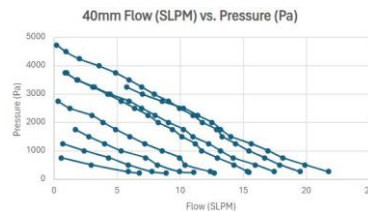
Stop

0 % Completion
 Successes
 Elapsed Time (s)

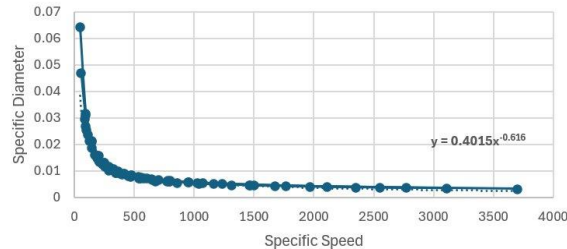
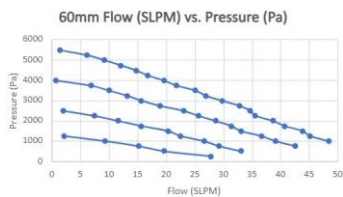
Pressure Rise	RPM	Rotor Diameter	Throat Diameter	Mass Flow Rate	Fluid Density

Model Features - Customizability

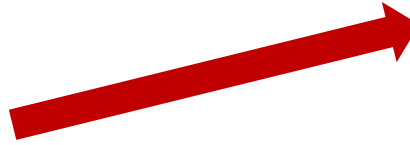
- Model Strength Based on Data
- Customizable tolerances, units, iteration step sizes



Ns Ds Diagram



..... Power (Ds)



MATLAB App

Partial Emissions Blower Model

Throat Diameter
Pressure Out
Diffuser
RPM
Pressure In
Rotor Diameter

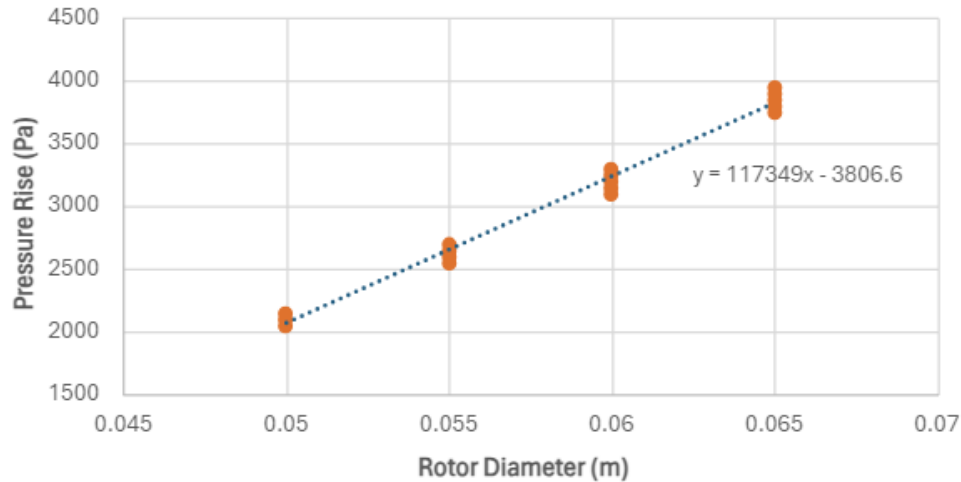
Mass Flow Rate Min: 0.0001 Max: 0.0008 Exact kgPerS
Rotor Diameter Min: 0.06 Max: 0.07 Exact m
Throat Diameter Min: 0.012 Max: 0.022 Exact m
Rotational Speed Min: 2.8e+0 Max: 3.5e+0 Exact RPM
Pressure Rise Min: 2500 Max: 2800 Exact Pa
Fluid: Air Iterations: 20448

● 100 % Completion
 Successes: 917
 Elapsed Time (s): 6.851

Pressure Rise (Pa)	Rotor Speed (RPM)	Rotor Diameter (m)	Throat Diameter (m)	Mass Flow Rate (kgPerS)	Fluid Density (kg/m3)
2500	28000	0.0600	0.0220	6.0000e-04	1.2250
2500	28000	0.0650	0.0170	5.0000e-04	1.2250
2500	28000	0.0650	0.0220	8.0000e-04	1.2250
2500	28100	0.0600	0.0220	6.0000e-04	1.2250
2500	28100	0.0650	0.0170	5.0000e-04	1.2250

Analysis and Trends - Pressure

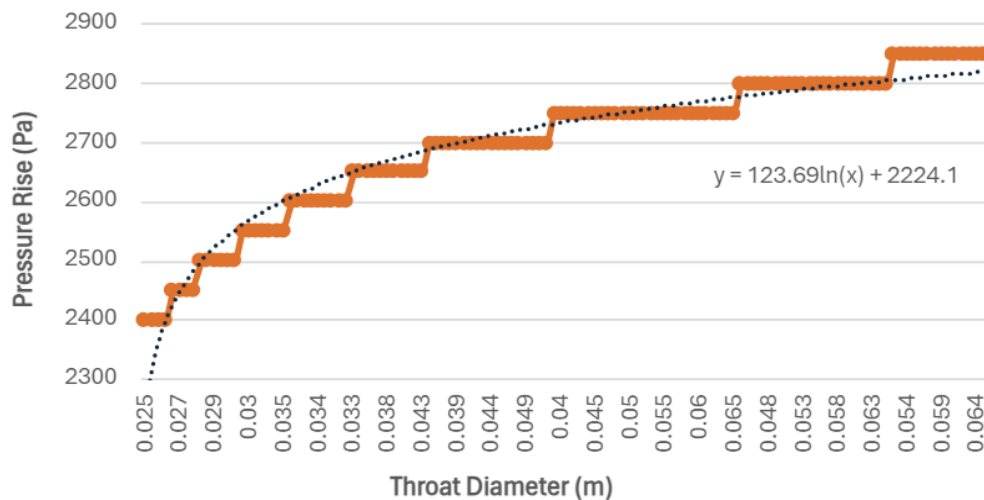
Rotor Diameter vs. Pressure Rise



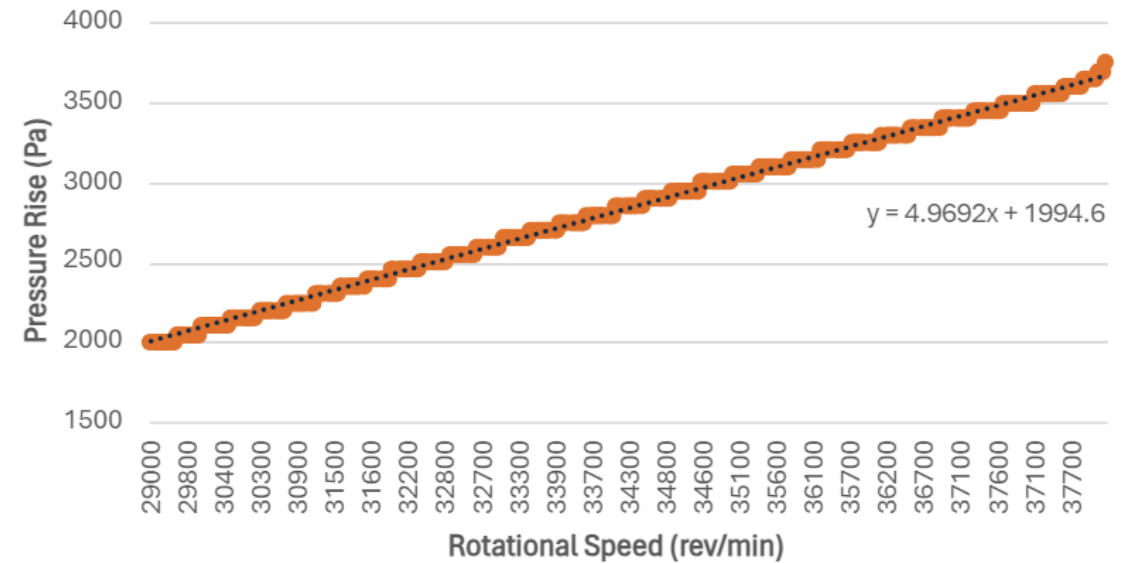
1. **Linear relationship** between rotor diameter, speed and pressure rise

2. **Slope becomes more steep** in relationship between throat diameter and pressure rise

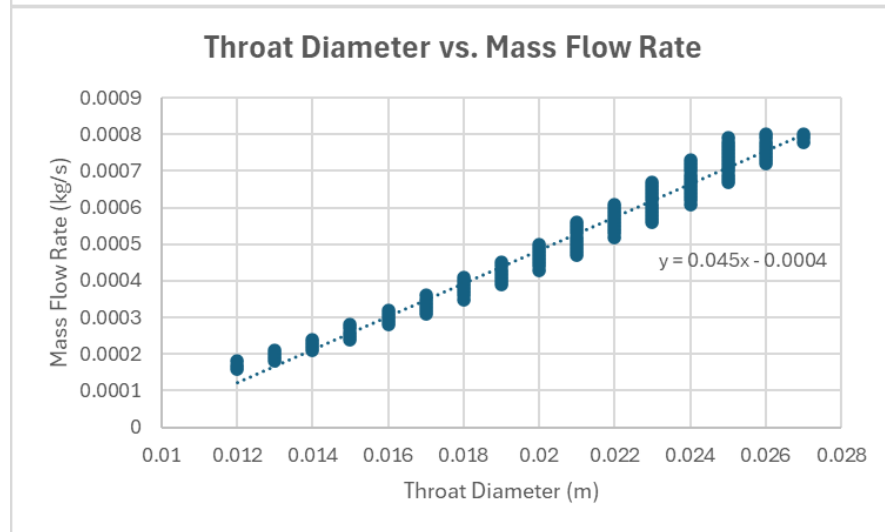
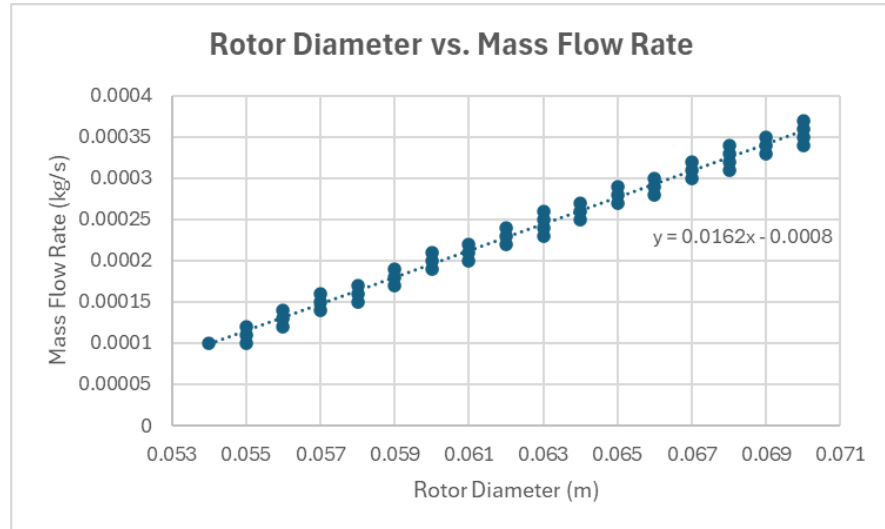
Throat Diameter vs. Pressure Rise



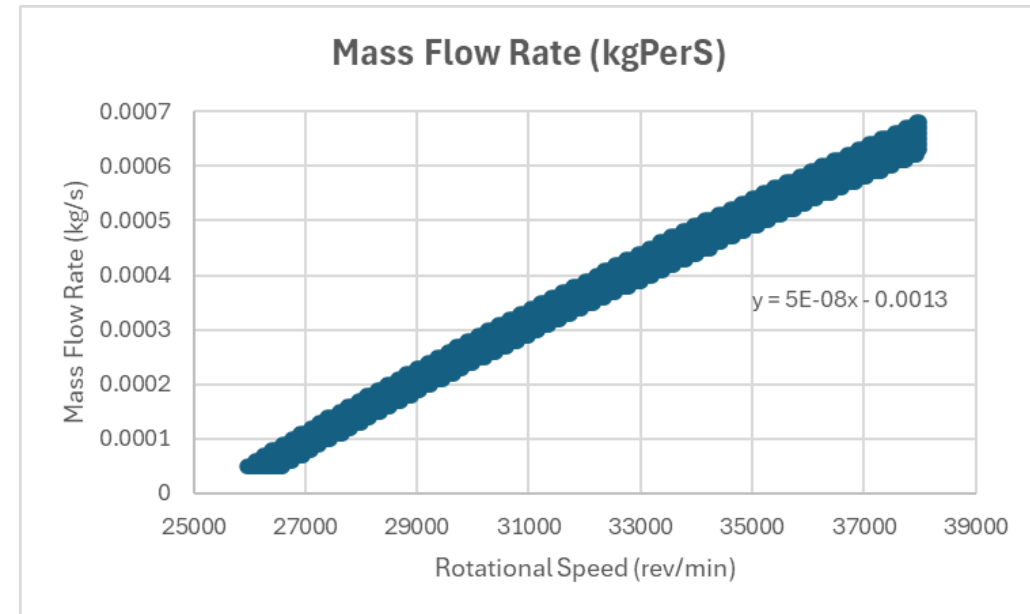
Rotational Speed vs. Pressure Rise



Analysis and Trends – Flow

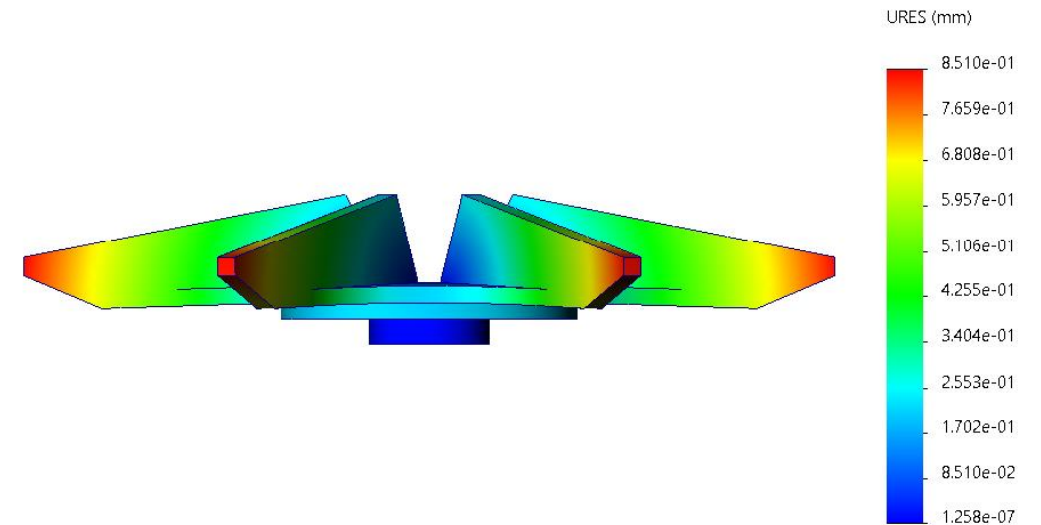
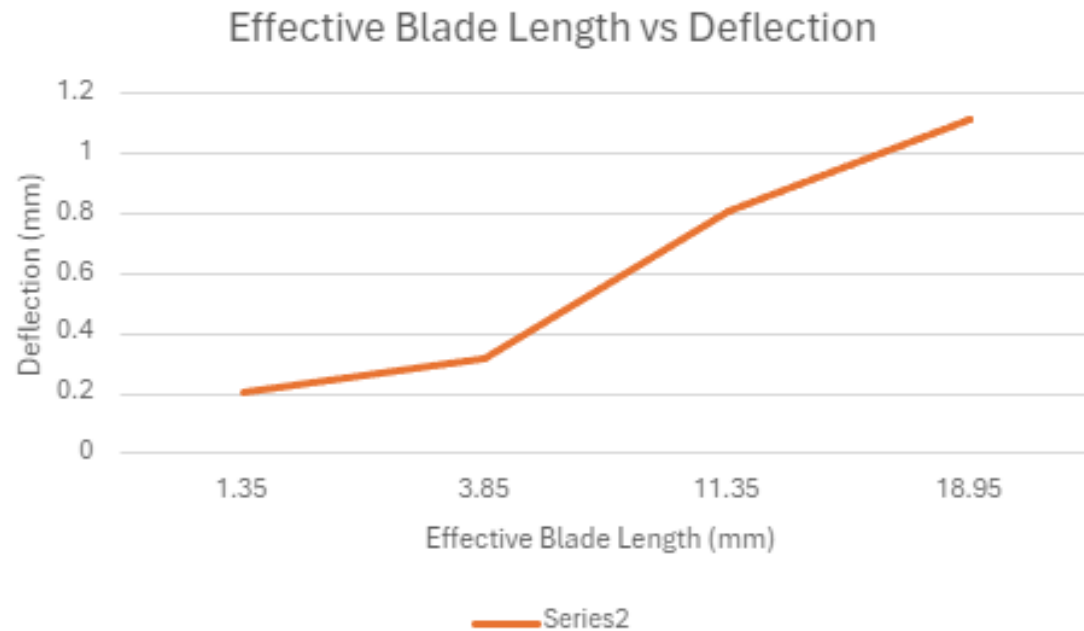


Linear relationship between diameters, speed and mass flow rate



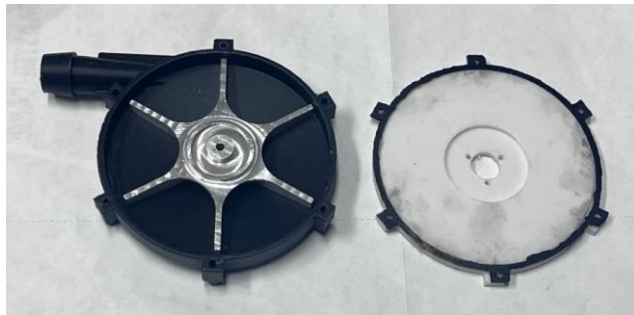
Deflection Results

- Found that the parameter that had the greatest effect on deflection was blade length
- Finalized on a redesign that would attempt to change the location of the max deflection felt rather than the overall value of it



Summary

- Primary Goal: Develop a tool that helps to explore several design parameters that affect pump characteristic outcomes and can suggest an effective pump design
 - Alternative to manufacturing pumps – reduced time in costs; manufacturing



Partial Emissions Blower Model

Mass Flow Rate Min 0.0001 Max 0.0008 Exact kgPerS

Rotor Diameter Min 0.06 Max 0.06 Exact m

Throat Diameter Min 0.012 Max 0.022 Exact m

Rotational Speed Min 2.8e+0 Max 3.5e+0 Exact RPM

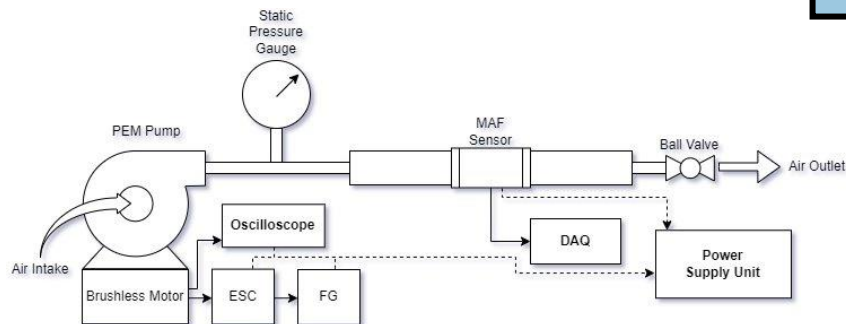
Pressure Rise Min 2500 Max 2800 Exact Pa

Fluid: Air Iterations: 8816

Calculate 100 % Completion

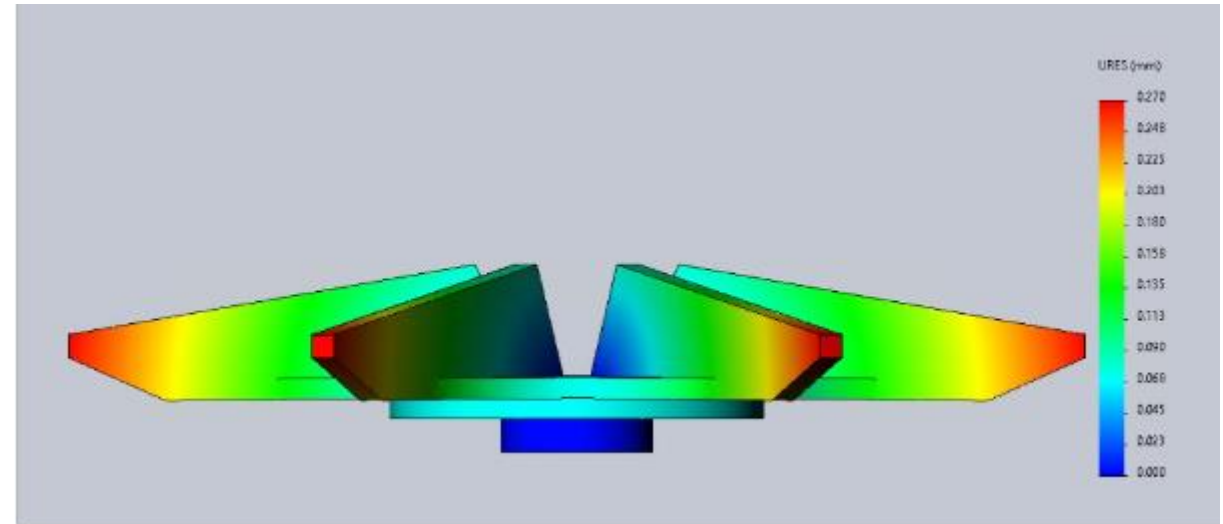
Stop Successes: 403 Elapsed Time (s): 6.187

Pressure Rise (Pa)	Rotor Speed (RPM)	Rotor Diameter (m)	Throat Diameter (m)	Mass Flow Rate (kgPerS)	Fluid Density (kg/m3)
2500	28000	0.0600	0.0220	6.0000e-04	1.2250
2500	28100	0.0600	0.0220	6.0000e-04	1.2250
2500	28200	0.0600	0.0220	6.0000e-04	1.2250
2500	28300	0.0600	0.0220	6.0000e-04	1.2250
2500	28400	0.0600	0.0120	2.0000e-04	1.2250



Summary

- Secondary Goals
 - Determine an effective method for manufacturing impellers that can be used at different scales
 - Determine an effective design change to mitigate blade deflection in non-metal impellers



Recommendations and Future Work

- Strengthen curve fits with 70mm data, additional data
- Look into other models to scale up pump
 - Geometrically similar, dissimilar pumps
 - Different diffuser geometry
- Test rotor deflection design change experimentally

The image features a large, bold, black text "Questions?" centered on the page. In the background, there is a faint, light gray watermark of the Worcester Polytechnic Institute logo. The logo is circular and contains the text "WORCESTER POLYTECHNIC INSTITUTE" around the top edge and "1865" at the bottom. In the center of the logo is a shield with a heart, flanked by laurel branches, and a banner above it that reads "LEHR UND KUNST".

Questions?

Sources

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PEM Diagram

